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# Experimental and computational investigation of flow through an annular turbine cascade

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# NAVAL POSTGRADUATE SCHOOL MONTEREY, CALIFORNIA



## THESIS

EXPERIMENTAL AND COMPUTATIONAL  
INVESTIGATION OF FLOW THROUGH  
AN ANNULAR TURBINE CASCADE

by

William Howard Donovan, Jr.

June 1995

Thesis Advisor:

Garth V. Hobson

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EXPERIMENTAL AND COMPUTATIONAL INVESTIGATION  
OF FLOW THROUGH AN ANNULAR TURBINE CASCADE

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## ABSTRACT

The purpose of this research was to compare numerical predictions with experimental results and to devise an accurate laser anemometry technique to measure in the endwall region of a confined annulus. Flowfield characteristics were computed using a three-dimensional flow solver with the numerical plane coincident with the experimental measurement location. An annular turbine cascade, designed for laser-Doppler velocimetry, was modified to obtain blade passage midspan surface pressure measurements. A range of predicted subsonic and transonic midspan surface pressures were compared favorably with experimental measurements. Two-dimensional Mach number, flow angle, and turbulence intensity measurements were obtained with a fiber-optics laser-Doppler velocimeter. The measurements were performed through a 1.0922 millimeter opening in the endwall at depths ranging from 0.01 mm to 3.34 mm and the results were compared with numerical predictions.



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## I. INTRODUCTION

Advanced aeropropulsion systems have required substantial technological improvements in turbomachinery. These depend on the availability of complex analysis and experimental tools. A thorough understanding of the generation and development of secondary flows in annular blade rows has provided insight into the design and performance of turbomachinery. Secondary flows, together with tip leakage flows, produce considerable flow distortions and losses in the endwall region. [Ref. 1] These losses can be minimized and turbomachinery efficiency can be improved with a more precise understanding of the flow mechanics and the ability to numerically predict the flow field. The current emphasis on turbomachinery design centers on numerical analysis.

This report specifically documents the investigation of the flow through an annular turbine cascade (ATC). References 2 through 4 include continuing research in the field of laser-Doppler velocimetry (LDV) and numerical prediction in a large annular turbine cascade. The authors provided experimental two and three-dimensional velocity, flow angle, and turbulence intensity at constant axial positions within an ATC passage. These measurements, in addition to vane surface static pressure measurements, supplied a test case for three-dimensional turbomachinery computer programs. They concentrated on obtaining comparisons ahead of, inside, and downstream of the blade passage and concluded that the largest difference between experimental and computational results was in the endwall region where viscous and secondary flow effects were the greatest.

Two previous investigations of the flow through the ATC have been conducted at the Naval Postgraduate School. Reference 5 included design and manufacturing information of the annular turbine cascade that was developed to determine the limitations of LDV measurements in a confined annulus. Reference 6 included additional laser and pressure probe access modifications and initial LDV measurements to the same ATC. This report includes further ATC modifications for midspan blade surface pressure measurements and LDV measurement techniques of the endwall flow. Radial two-dimensional fiber-optic probe traverses were performed, through a small access hole in the outer casing, to coincide with the numerical exit plane. Circumferential surveys were obtained at different radial locations close to the endwall. Blade midspan surface pressures were measured within one blade passage at various inlet total-to-downstream hub-static pressure ratios. Blade surface pressure and endwall flow measurements were compared with numerical predictions obtained using a three-dimensional viscous computer program.

The two most noteworthy comparisons were at a subsonic pressure ratio and a pressure ratio corresponding to sonic exit conditions. Comparisons with the LDV data were performed at the subsonic flow condition. The numerical blade surface pressure distributions compared well with the experimental results, particularly for the sonic exit condition for which trailing edge shocks were predicted.

## II. EXPERIMENTAL SETUP

### A. TEST FACILITY AND ANNULAR TURBINE CASCADE

Airflow for the annular turbine cascade experiment was provided by a VA-312 Allis-Chalmers 12-stage axial-flow compressor located at the Turbopropulsion Laboratory of the Naval Postgraduate School. The compressor was operated at 12,000 rpm at various discharge pressures and provided metered air to a plenum chamber. Air from the plenum was routed to a 232.918 mm (9.170 in) diameter bellmouth and test section through honeycomb flow straighteners in a 254 mm (10 in) flanged steel pipe as shown in Figures 1 and 2.

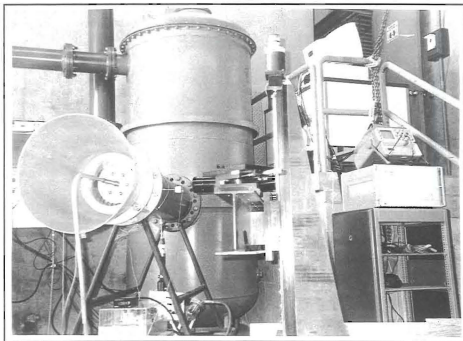


Figure 1. Front View of Experimental Apparatus

Flow stagnation pressure was measured at two upstream locations. One combination probe provided pressure-setting information to a mercury manometer board and a digital readout of flow stagnation temperature, while the second probe was connected to a Scanivalve (Figure 2, lower right). Four (averaged) upstream static ports and four (averaged) inner hub



downstream (one-half axial chord) static ports were also connected to a Scanivalve for automated pressure data acquisition.

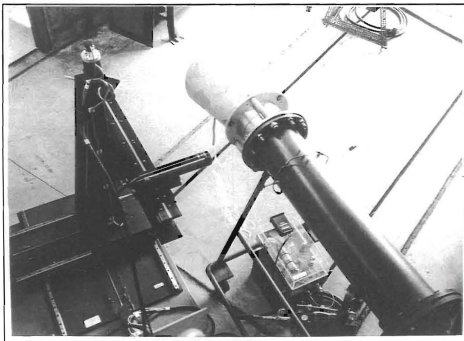


Figure 2. Top View of Experimental Apparatus

Atomized glycerin particles constituted the LDV seed material which were introduced through a 7.938 mm (0.313 in) diameter copper tube approximately 108 tube diameters upstream of the test section. Seed atomization was performed using a commercial TSI, Inc., Six-Jet Atomizer connected to the laboratory compressed air supply. Laser traverse mechanism buffet, caused by the exiting flow, was minimized with a prefabricated aluminum cone (Figure 2, upper center) and Reference 6 showed that flow characteristics were unaltered with the cone attached.

Each blade was designed with a combination of simple circular arcs and line segments and included a leading edge radius of 2.413 mm (0.095 in), trailing edge radius of 0.305 mm (0.012 in), and axial chord of 24.77 mm (0.975 in). The annular stator row was manufactured from 2218-T61 aluminum and consisted of 31 blades with a midspan spacing

of 21.77 mm (0.857 in), resulting in a blade solidity of 1.14. The inner hub radius was 98.93 mm (3.895 in) and the outer case radius was 116.46 mm (4.585 in) with the same profile at all radii. Reference 5 included the original set of manufacturing drawings and Reference 6 included a description of the wake positioning system.

## B. PRESSURE MEASUREMENT MODIFICATIONS

The original ATC did not provide the capability to measure blade surface pressures. Within a single passage, seven suction-side static ports and four pressure-side static ports were drilled orthogonal to the blade surface at midspan. Figures 3 and 4 show front and rear views of the instrumented blades. As shown in Figure 5, each static port was 0.406 mm (0.016 in) in diameter and each spanwise hole was 1.321 mm (0.052 in) in diameter.

All spanwise holes were sealed at the tips and stainless steel tubes were cemented into the hub openings. The tubes were connected to the Scanivalve with plastic tubing which was fed out through a sting from the center body. (Figures 1 and 2) Each port's circumferential position was measured using a 2.375 mm (0.0935 in) diameter stylus and later converted graphically to an axial chord position to allow computational comparisons. Figure 5 shows port numbering and dimensions.



Figure 3. Blade Leading Edge View With Pressure Measurement Modifications

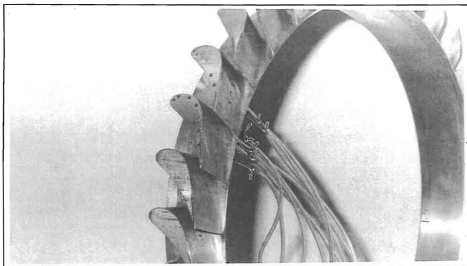


Figure 4. Blade Trailing Edge View With Pressure Measurement Modifications

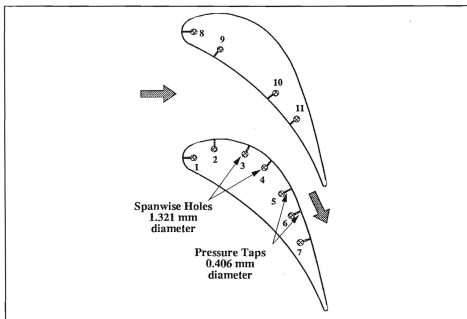


Figure 5. Blade Static Port Numbering Sequence And Dimensions

### C. PRESSURE DATA ACQUISITION

The data acquisition system, for the pressure measurements, is shown schematically in Figure 6. All data acquisition was remotely controlled by a Hewlett-Packard 9000 computer system. Appendix A contains the program utilized to conduct all pressure data acquisition. A Scanivalve was connected to a Model HG-78K Scanivalve controller, which in turn was connected to a Hewlett-Packard, Model 3456A Digital Voltmeter and Model 3495A Scanner via a HP-IB instrument bus. Scanivalve calibration was performed to within an accuracy of  $\pm 0.1$  inches mercury. Table A1 in Appendix A relates each Scanivalve port to its respective pressure measurement.

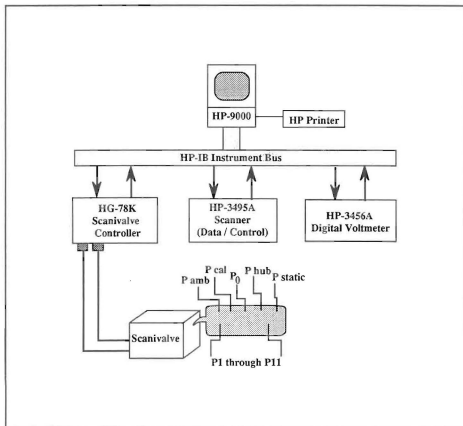


Figure 6. Pressure Data Acquisition Schematic

#### D. LASER-DOPPLER VELOCIMETER

Part of the laser apparatus is shown in Figure 7. The probe, processor, computer, and traverse mechanism are shown in Figures 1 and 2. The LDV system is shown schematically in Figure 8. A LEXEL Model 95 four-Watt argon-ion laser was connected to a TSI, Inc., Model 9201 ColorBurst multicolor beam separator. The beam separator divided the incoming light into shifted and unshifted beams, with the shifted beam receiving a 40 MHz frequency shift from a Bragg cell. The two beams were further split into three polarized pairs: green (514.5 nm), blue (488 nm), and violet (476.5 nm).

Individual couplers on the ColorBurst directed each beam to the laser probe via a fiber-optic cable. Each fiber-optic probe contained receiving optics which directed the return signal to a TSI, Inc., Model 9230 ColorLink multicolor receiver. The ColorLink provided photomultiplier and frequency-shifting functions. All conditioned ColorLink signals were sent to a TSI, Inc., IFA-750 digital burst correlator where valid Doppler signals were identified and digitized.

The fiber-optic probes were mounted to a LINTECH, Model 41583 traverse table. An Applied Motion Products System 1618 traverse controller was used manually to control traverse table movement. All ColorLink and IFA-750 functions and LDV data processing were accomplished remotely by computer using TSI's menu-driven software, FIND (FLOW INFORMATION DISPLAY) version 4.04.

#### E. EXPERIMENTAL PROCEDURE

##### 1. Midspan Surface Pressure Measurements

Midspan surface pressure measurements were obtained with the pressure data acquisition system. The pressure ratio ( $P_{rat}$ ) was defined as the downstream hub-static pressure ( $P_{hub}$ ) divided by the upstream stagnation pressure ( $P_0$ ). Each pressure ratio was set by metering the upstream stagnation pressure until a desired mercury manometer column height was achieved. Five pressure ratios (0.5070, 0.6041, 0.6815, 0.8077, and 0.9054) were considered and during each run all the blade surface pressures,  $P_0$ , and  $P_{hub}$  were recorded.

##### 2. Laser Alignment

LDV alignment for endwall flow measurements was accomplished as shown in Figure 9. The objective of the LDV alignment procedure was to center the probe volume in the 1.0922 mm optical access hole at a known and repeatable radial distance. The two-dimensional fiber-optic probe was attached to a mounting-bracket micrometer which allowed

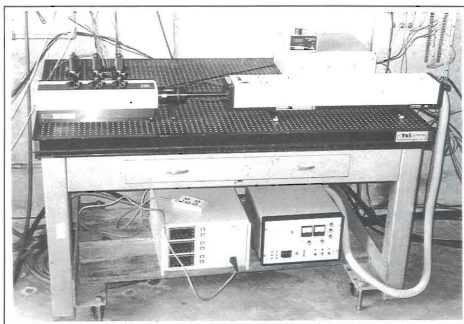


Figure 7. LDV Bread Board (with Laser, Color Separator, and Receiving Optics Module)

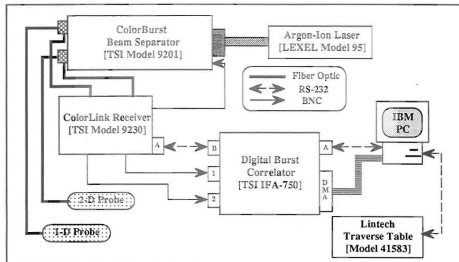


Figure 8. LDV System Schematic

probe radial travel in 0.01 mm increments. After zeroing the micrometer, the traverse table was manually advanced forward until the four beam separation was minimized, yet discernible with the naked eye (Figure 9, sketch A). The digital position-indicator reading on the traverse controller was noted and the process repeated for forward travel until the same image reappeared. The midpoint of the traverse table positions was defined as the center of the probe volume with the face of the optical access plate as a radial reference point. Reference 6 described complete dimensions of the optical access plate.

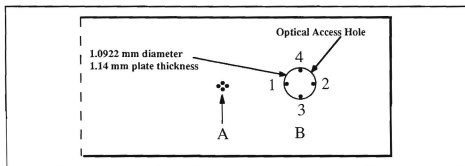


Figure 9. LDV Alignment Schematic

Horizontal and vertical alignment positions (Figure 9, sketch B) were obtained by noting the digital position indicator on the traverse controller as the probe volume touched the left(1), right(2), bottom(3), and top(4) inner edge of the optical access hole. The horizontal and vertical reference positions were defined as the center of the optical access hole. The probe was then traversed inward by 1.14 mm (the thickness of the optical access plate), at which point the center of the probe volume coincided with the outer (case) wall of the turbine cascade. This alignment technique was repeatable and ensured that the probe volume passed cleanly through the center of the optical access hole.

### 3. Endwall Measurements

All endwall measurements were conducted at a pressure ratio of 0.9054. The laser beams were aligned with the downstream optical access hole as described above, and then traversed manually using the mounting-bracket micrometer. Radial endwall surveys at 0.01, 0.06, 0.18, 0.42, 0.89, 1.78, and 3.34 millimeters from the tip casing were conducted for peripheral (wake) angular settings ranging from -8 degrees to +8 degrees. One degree (wake) position increments were achieved by circumferentially rotating the blade row and

center body within the outer casing. At each circumferential position the inner section was secured in place with a locking bolt arrangement, the design of which was documented in Reference 6. Seeding position, ColorLink, and IFA-750 settings were adjusted for an optimum LDV data rate and minimum noise. All settings ensured a minimum data rate of 100 samples per second, however; LDV data rate was extremely sensitive to position of the wand which introduced seed into the flow.

A 'random' mode processor setting allowed a total of 1,024 samples between the green and blue channels with no user control over sample distribution. The 'coincidence' mode acquired 1,024 samples for each channel. Repeatability measurements for endwall velocity, flow angle, and turbulence intensity were separately conducted at three wake positions for both modes. A minimum pressure ratio was determined before data rate conditions became unacceptable. Raw data were converted within FIND and manually transferred to a spreadsheet for further processing.





### III. COMPUTATIONAL FLUID DYNAMICS

#### A. GRID GENERATION

Grid generation was completed using the FORTRAN language program "Turbomachinery C GRID (TCGRID)" [Ref. 7]. TCGRID inputs consisted of four lines of namelist inputs followed by a title, hub and tip geometry, and blade geometry. The blade geometry was input in cylindrical coordinates ( $z$ ,  $\theta$ ,  $r$ ) starting at the blade trailing edge and wrapping clockwise. The blade inputs were completed in stacked sections from the hub to the tip. The grid used, with resolution of  $150 \times 31 \times 65$  ( $i, j, k$ ), was the same as that generated in Reference 6. The  $i$ -index was defined clockwise from the lower (pressure surface) exit to the upper (suction surface) exit, the  $j$ -index was defined from the blade surface to the periodic boundary, and the  $k$ -index was defined from the hub to the tip. All computational solutions were based on this grid. The final grid is shown in Figure 10. Appendix B contains the grid namelist input file and Figure B1 shows the blade geometry.

#### B. COMPUTATIONAL SCHEME

Flowfield solutions were obtained using two versions (206 and 208) of "Rotor Viscous Code 3-D (RVC3D)", a FORTRAN language program designed for analysis of three-dimensional viscous flows in turbomachinery. RVC3D was written to solve the thin-layer Navier-Stokes equations in Cartesian coordinates. The equations were discretized using second-order finite-differences in space and solved in time with a fourth-order Runge-Kutta scheme. Streamwise viscous terms were neglected using a thin-layer assumption, but cross-channel viscous terms were retained. A spatially-varying time step and implicit residual smoothing were used to accelerate convergence. [Refs. 8-10] Turbulence effects were modeled using a 3-D adaptation of the Baldwin-Lomax model and the Cebeci-Smith model.

RVC3D version 206 was used to predict the flowfield for pressure ratios of 0.6041, 0.6815, 0.8077, and 0.9054. The turbulence in the flowfield was computed with an adaptation of the Cebeci-Smith turbulence model. This version of RVC3D only accommodated subsonic exit boundary conditions whereby the hub static pressure was held at the pressure ratio and radial equilibrium was solved for the spanwise pressure distribution. Version 208 of RVC3D was used for the 0.5070 pressure ratio, which produced trailing edge shocks that extended to the exit plane. This newer version allowed for a supersonic exit boundary condition where the exit conditions were based on Giles' characteristic boundary conditions [Ref. 11]. The Baldwin-Lomax turbulence model was used for this test case since

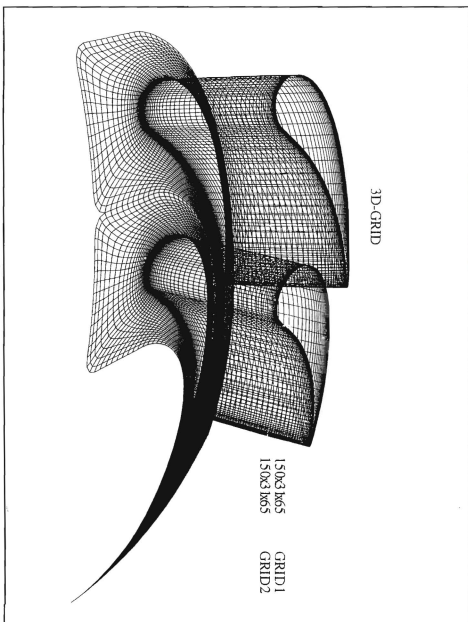


Figure 10. Three-Dimensional Multiple Grid (150 x 31 x 65)

a 'bug' was discovered that would not allow the code to properly recognize the Cebeci-Smith turbulence model [Ref. 12]. The code's author has since corrected the program.

Appendix C contains an example namelist input file used to obtain a flow solution and a description of the steps required to run the code on the Naval Postgraduate School (NPS) and National Aerodynamic Simulation Program (NAS) Cray supercomputers. Output solution file (fort.3) information was visually examined with FAST and PLOT3D graphics software [Refs. 13 and 14]. Solution residual files (fort.4), blade surface pressure files (fort.7), and Mach number and flow angle files (fort. 7-11) were calculated from the solution file with the FORTRAN programs "pxy.f" and "plane.f" (Appendix G) for final graphical representation.



## IV. RESULTS AND DISCUSSION

### A. BLADE MIDSPAN SURFACE PRESSURE MEASUREMENTS

Experimental blade-surface pressure measurements were averaged and a maximum repeatability error was calculated. Each blade static port reading was non-dimensionalized by the upstream stagnation pressure ( $P_0$ ). In order to provide an experimental comparison with Reference 6 for the 0.6815 pressure ratio, experimental repeatability was confirmed by conducting six runs; three runs each on separate days (Tables D1 and D2 of Appendix D). The maximum repeatability error was 0.87%. All other mean pressure data were based on a three run average at each pressure ratio and resulted in a maximum repeatability error ranging from 0.66% at the lowest pressure ratio down to 0.22% at the highest pressure ratio. Table 1 contains each midspan static port location and non-dimensional pressure measurement for all experimental pressure ratios. Tables D3 through D6 of Appendix D contain pressure data for pressure ratios of 0.5070, 0.6041, 0.8077, and 0.9054 respectively.

Static Port Number	Fraction of Axial Chord (x/c)	Pressure Ratio = 0.5070	Pressure Ratio = 0.6041	Pressure Ratio = 0.6815	Pressure Ratio = 0.8077	Pressure Ratio = 0.9054
1	0.0000	1.0116	1.0019	0.9999	0.9996	1.0012
2	0.1322	0.9628	0.9576	0.9600	0.9729	0.9872
3	0.4615	0.8613	0.8646	0.8791	0.9171	0.9566
4	0.6604	0.6986	0.7223	0.7595	0.8433	0.9195
5	0.7757	0.5419	0.6216	0.6910	0.8113	0.9057
6	0.8542	0.5584	0.6358	0.7031	0.8190	0.9103
7	0.9183	0.5200	0.6166	0.6928	0.8169	0.9102
8	0.0000	1.0082	0.9987	0.9992	0.9993	1.0004
9	0.1322	0.9955	0.9878	0.9893	0.9923	0.9973
10	0.6274	0.9389	0.9362	0.9439	0.9600	0.9810
11	0.7508	0.8853	0.8874	0.9015	0.9327	0.9668

Table 1. Non-Dimensional Midspan Surface Pressure ( $P/P_0$ )

### B. LASER-DOPPLER VELOCIMETRY MEASUREMENTS

Two-dimensional LDV measurements were obtained to an approximate depth of 3.34 mm through a 1.0922 mm (0.043 in) diameter hole located one-half axial chord downstream. All LDV data were acquired at a pressure ratio of 0.9054 and are tabulated in

Tables E1 through E5 of Appendix E. All data resulted from programmed editing of histograms by the FIND software and data outside two standard deviations were discarded.

The axial velocity ( $V_z$ ) and tangential velocity ( $V_{\theta}$ ) were combined to form the total velocity ( $V_{\text{total}}$ ) as shown in Figure 11. Flow velocities are seen to decrease toward the case wall. The boundary layer was distorted due to secondary flows and wake and corner vortices that form within the blade passage. Two-dimensional endwall flow angle was defined as the arc tangent of the tangential velocity divided by the axial velocity and is shown graphically in Figure 12. Periodicity is evident over 11.6 degrees (31 blades).

Turbulence intensities were calculated with respect to the maximum downstream exit velocity ( $V_{\text{exit}}$ ). Figures 13 and 14 show turbulence intensity in both the tangential and axial directions. The tangential turbulence intensity is seen to be higher, possibly due to the steep gradient of the  $V_{\theta}$  mean-flow profile in the radial direction.

The 0.9054 pressure ratio provided excellent seeding conditions and ATC vibrations were low. Data rates ranged from approximately 300 samples per second at the 3.34 mm depth to 150 samples per second at the 0.01 mm depth. Seeding material slowly accumulated inside the lower portion of the optical access hole and interfered with LDV data acquisition. Occasionally, the atomizer was secured and seed material allowed to disperse. Hole alignment and laser power (1.5 Watts) were periodically verified as these could drift due to temperature changes. Endwall measurement techniques did not provide pressure equalization across the optical access hole. During one exploratory run a minimum pressure ratio of 0.80 was achieved before the data rates deteriorated to unacceptable levels.

A random mode comparison resulted in an average repeatability difference of 2.9%, 0.6%, 6.4%, and 7.0% for velocity, flow angle, tangential turbulence intensity, and axial turbulence intensity respectively. Figures 15 through 23 graphically depict velocity, flow angle, and turbulence intensity random mode repeatability data for wake positions of +7, 0, and -8 degrees. A random and coincidence mode comparison resulted in an average maximum difference of 0.4%, 0.2%, 4.1%, and 6.3% for velocity, flow angle, tangential turbulence intensity, and axial turbulence intensity respectively. Tables F1 and F2 of Appendix F contain LDV repeatability data.

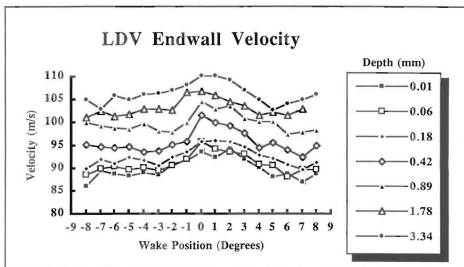


Figure 11. Two Component LDV Endwall Velocity

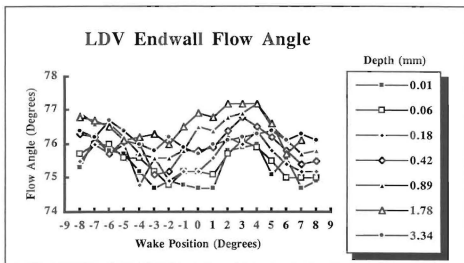


Figure 12. Two Component LDV Endwall Flow Angle



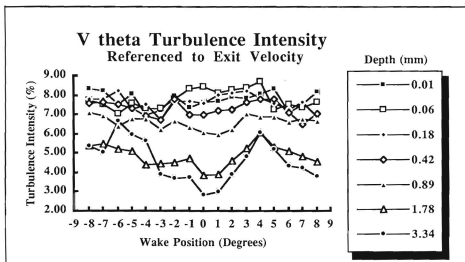


Figure 13. Two Component LDV Endwall Turbulence,  $V_{\theta}$

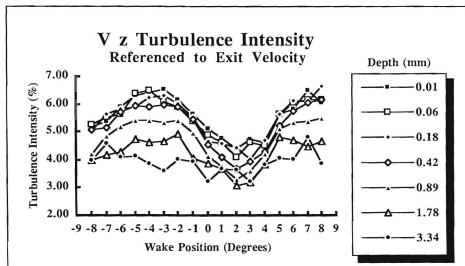


Figure 14. Two Component LDV Endwall Turbulence,  $V_z$

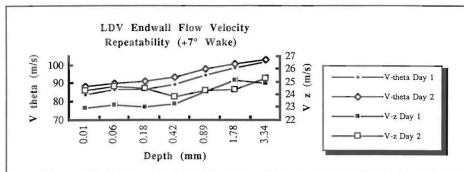


Figure 15. LDV Endwall Flow Velocity Repeatability At +7 Degrees Wake Position

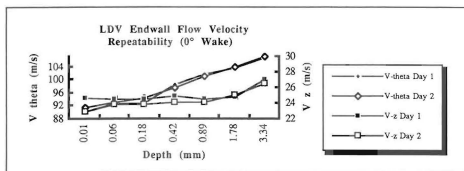


Figure 16. LDV Endwall Flow Velocity Repeatability At 0 Degrees Wake Position

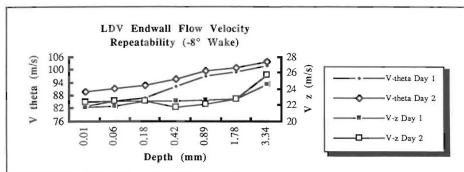


Figure 17. LDV Endwall Flow Velocity Repeatability At -8 Degrees Wake Position

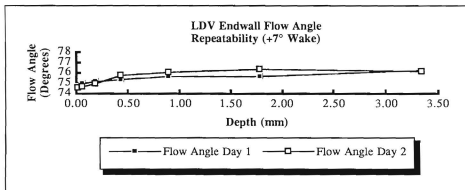


Figure 18. LDV Endwall Flow Angle Repeatability At +7 Degrees Wake Position

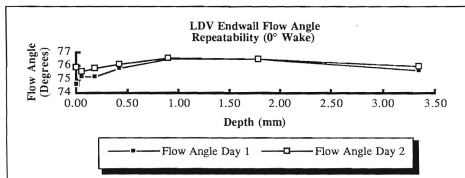


Figure 19. LDV Endwall Flow Angle Repeatability At 0 Degrees Wake Position

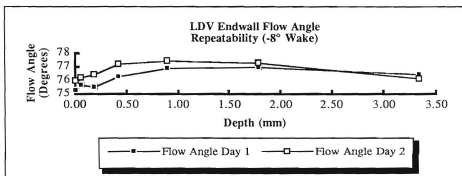


Figure 20. LDV Endwall Flow Angle Repeatability At -8 Degrees Wake Position

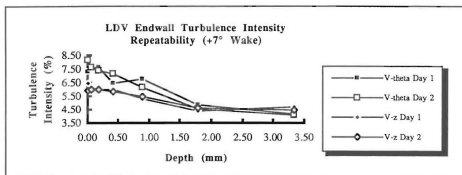


Figure 21. LDV Endwall Turbulence Repeatability At +7 Degrees Wake Position

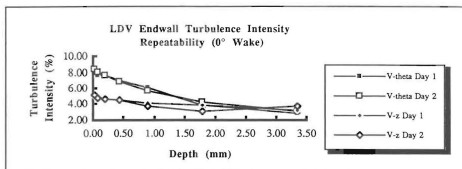


Figure 22 LDV Endwall Turbulence Repeatability At 0 Degrees Wake Position

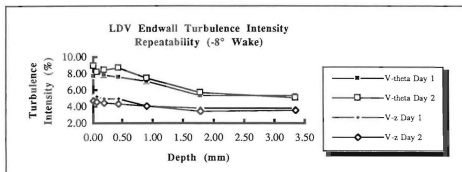


Figure 23. LDV Endwall Turbulence Repeatability At -8 Degrees Wake Position

## C. NUMERICAL COMPARISON

Computational Fluid Dynamic (CFD) solutions were run with the same pressure ratios used in the experiments. Appendix G briefly describes the computational data reduction process and programs utilized. Graphical output of the numerical results was useful in obtaining a qualitative understanding of the flow characteristics.

### 1. Blade Midspan Surface Pressures

Reference 6 predicted hub, midspan and tip surface pressures at a pressure ratio of 0.68. Figure 24 shows a comparison between numerical and experimental blade surface pressures at midspan for a pressure ratio of 0.6815. The suction surface curve suggests that the position of the blade passage throat was at 0.8 of axial chord and, in fact, the throat was located between static ports five and six. The minimum and maximum values at static ports five, six, and seven may be due to the boundary layer interaction at the blade's blunt trailing edge. Figures 25 through 28 show a comparison between numerical and experimental blade midspan surface pressures at pressure ratios of 0.5070, 0.6041, 0.8077 and 0.9054 respectively. Comparison of the blade surface measurements and the computational results generally show excellent agreement.

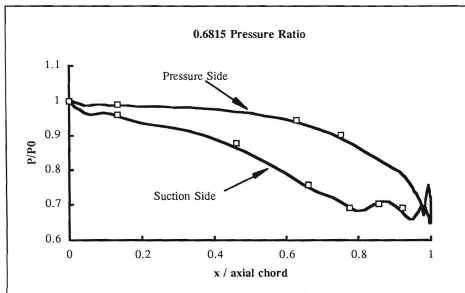


Figure 24.  $P/P_0$  vs.  $x/c$  for 0.6815 Pressure Ratio

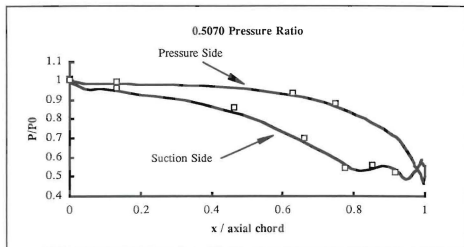


Figure 25.  $P/P_0$  vs.  $x/c$  for 0.5070 Pressure Ratio

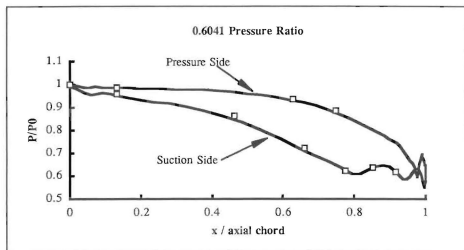


Figure 26  $P/P_0$  vs.  $x/c$  for 0.6041 Pressure Ratio

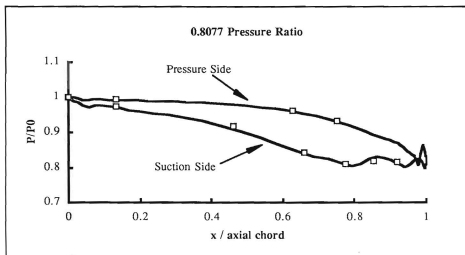


Figure 27.  $P/P_0$  vs.  $x/c$  for 0.8077 Pressure Ratio

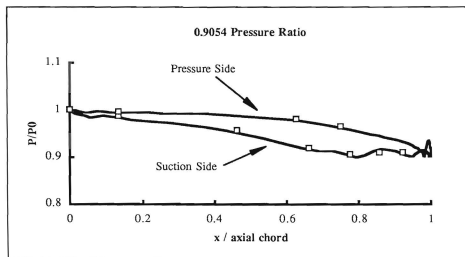


Figure 28.  $P/P_0$  vs.  $x/c$  for 0.9054 Pressure Ratio

## 2. Mach Number and Flow Angle

Two-dimensional Mach number and flow angle comparisons were made between numerical and experimental results for the surveys at each of the seven radial positions. Figures 29 through 32 compare Mach number, while Figures 33 through 36 compare flow angle at four selected radial positions. The remaining comparisons are included in Appendix H. All comparisons were based on a circumferential match between the experiment and computation, by shifting the latter to coincide with the measured profile. The specific matching was based on the maximum Mach number at the deepest radial position (3.34 mm). This matching was kept constant for the circumferential comparisons at all other spanwise locations. To depict flow periodicity and equivalently compare with experimental wake positions, the numerical solutions were repeated over one and one-half blade passages.

Computed Mach numbers differed from the experimental values by an average 12.7% between the surveys at 0.18 mm and 3.34 mm. The difference increased to 70.8% near the endwall, possibly due to a combination of LDV data velocity biasing and insufficient grid resolution. Velocity biasing was estimated to cause an approximate 11.0% velocity increase in the endwall region. The numerical solution at the endwall (0.01 mm) was based on the last k grid point in the radial direction. Computed flow angles surprisingly only varied from the experimental values by an average 3.2% (2.3 degrees). This was in contrast to the measurements in Reference 6 ( $P_{rat} = 0.68$ ) which showed poor comparison between measured and computed flow angles across the wake at 90 percent span.

The question of radial spatial error with the probe volume needed to be determined. The probe volume length and diameter (Figure 37) was calculated as 1.56 mm and 0.11 mm respectively. The probe volume had approximately 30 fringes across its minor axis and after data processing an average of 13 fringe crossings constituted a valid Doppler burst. Since the IFA-750 digital burst correlator automatically centered the valid signal within the probe volume, an effective probe volume with length 1.39 mm and diameter 0.04 mm was obtained. Figure 38 depicts, to scale, the radial survey resolution that results from endwall depth and effective probe volume dimensions. The decrease in relative spatial resolution close to the endwall also contributed to the lack of agreement between the Mach number profiles (Figure 32).



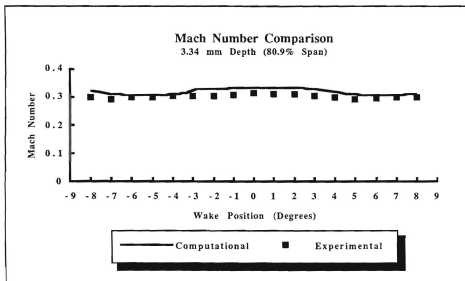


Figure 29. Mach Number Comparison At 3.34 mm Depth

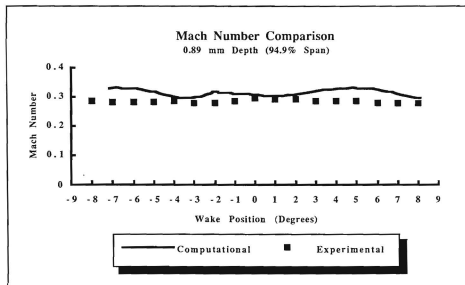


Figure 30. Mach Number Comparison At 0.89 mm Depth

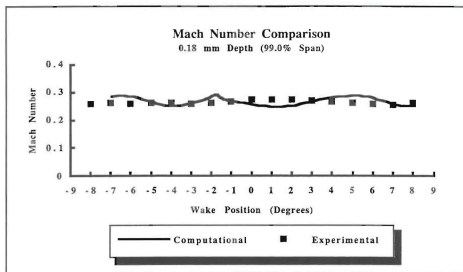


Figure 31. Mach Number Comparison At 0.18 mm Depth

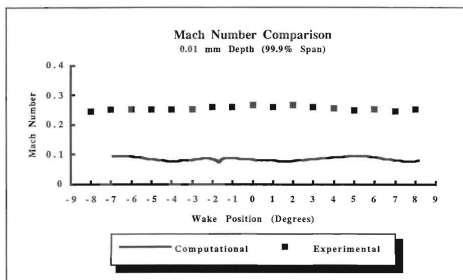


Figure 32. Mach Number Comparison At 0.01 mm Depth

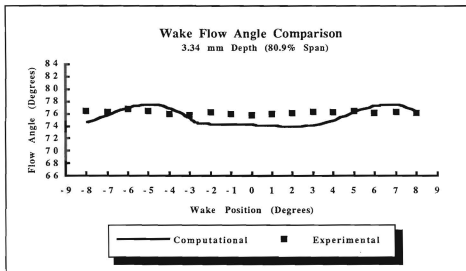


Figure 33. Wake Flow Angle Comparison At 3.34 mm Depth

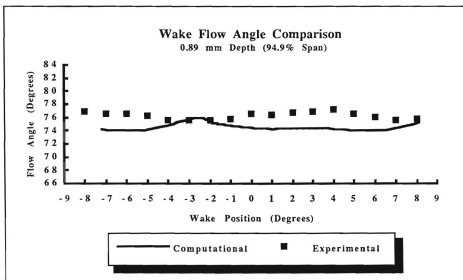


Figure 34. Wake Flow Angle Comparison At 0.89 mm Depth

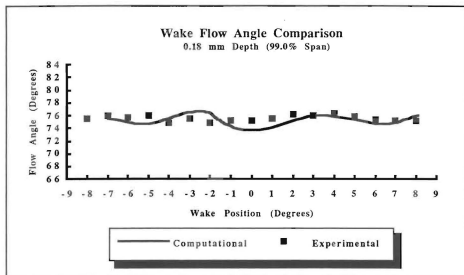


Figure 35. Wake Flow Angle Comparison At 0.18 mm Depth

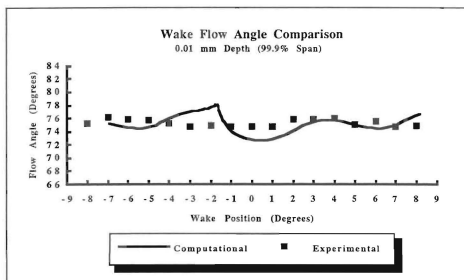


Figure 36. Wake Flow Angle Comparison At 0.01 mm Depth

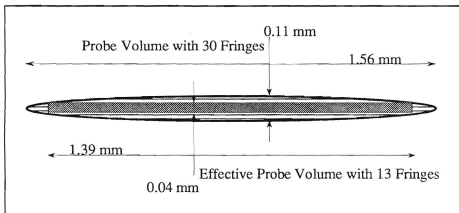


Figure 37. Probe Volume Dimensions

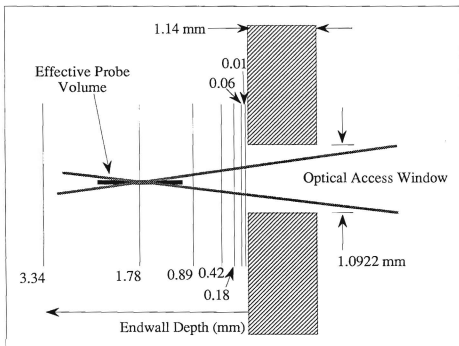


Figure 38. Endwall Measurement Schematic

## D. NUMERICAL RESULTS

### 1. Convergence History

Solutions at pressure ratios of 0.5070, 0.6041 and 0.6815 fully converged after approximately 1,200 iterations. Figure 39 shows density residuals at 0.6815 pressure ratio after 10,200 iterations. Solutions for pressure ratios of 0.8077 and 0.9054 were initially obtained after 1,200 iterations, but convergence did not occur (i.e. three orders of magnitude reduction of the residuals) until approximately 3,000 iterations. Figures 11 through 15 of Appendix I contain density-residual convergence history for all examined pressure ratios.

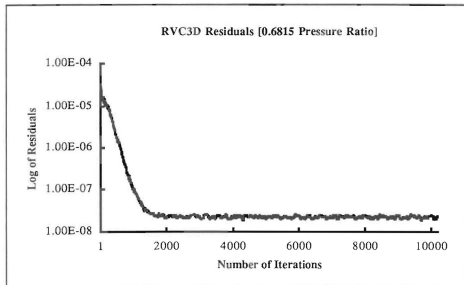


Figure 39. 0.6815 Pressure Ratio Convergence History (10,200 Iterations)

### 2. Turbulence Models

At a pressure ratio of 0.8077, the code was run with the following turbulence models;

- RVC3D version 206 Cebeci-Smith
- RVC3D version 206 Baldwin-Lomax
- RVC3D version 208 Baldwin-Lomax

so as to assess the effect of different turbulence models on the solution. In all three cases, the code produced similar solutions.

### 3. Mach Number Contours

Figures 40 and 41 display exit plane Mach number contours at pressure ratios of 0.9054 and 0.5070 respectively. The Mach number contours between 25% and 75% span provide an indication of where the flow characteristics were well established. Boundary layers appear, as expected, to be thin near the endwall region and thicker near the hub. Spanwise wake curvature, due to secondary flows, is also apparent in the two figures with Figure 40 showing the greater curvature.

Figure 42 displays midspan Mach number contours for the blade passage at 0.5070 pressure ratio. The flow was strongly accelerated along the suction side. The acceleration of the flow to high local Mach numbers resulted in shocks at the throat and trailing edge. The wake was a merging of the pressure and suction side boundary layers. The trailing-edge shock extended across the wake from the adjacent (upper) blade, out to the exit plane of the grid. The shock interaction with the wake caused the wake to narrow down, resulting in a wake which was repeatedly diffused and then coalesced. The shock strength decreased away from the trailing edge as a result of this interaction with the wake, which could be both physical and computational in nature. Because of the relatively good agreement between experiment and computation of the blade surface pressure field, it was felt that the solution was realistic. Appendix J includes program listings for a graphical display of multiple grids and corresponding numerical solutions for use with PLOT3D.

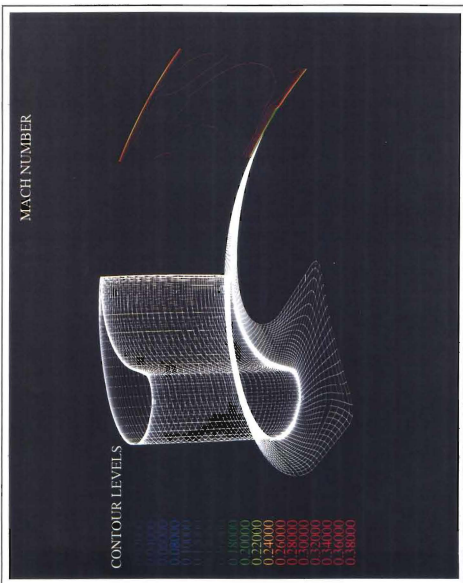


Figure 40. Exit Plane Mach Contours (0.9054 Pressure Ratio)





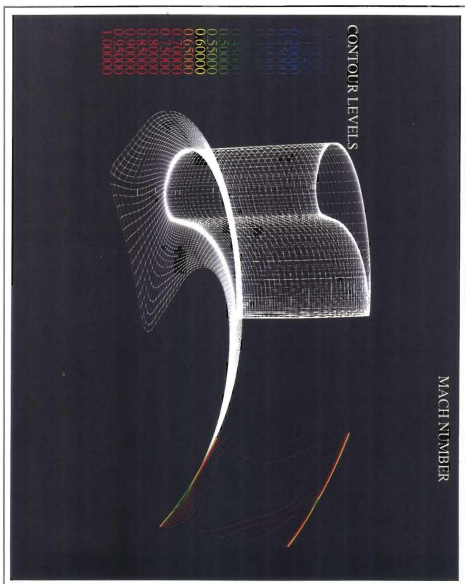


Figure 41. Exit Plane Mach Contours (0.5070 Pressure Ratio)



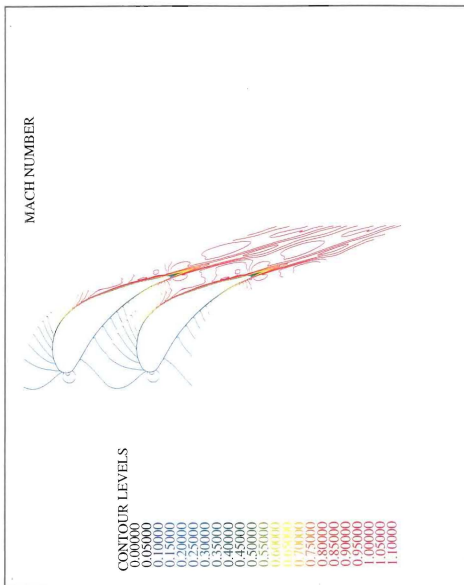


Figure 42. Midspan Mach Contours (0.5070 Pressure Ratio)







## V. CONCLUSIONS AND RECOMMENDATIONS

An experimental and computational investigation was conducted of the flow through an annular turbine cascade. Computational pressure ratios ( $P_{rat}$ ), defined as the downstream hub-static pressure ( $P_{hub}$ ) divided by the upstream stagnation pressure ( $P_0$ ), were chosen to coincide with experimental test conditions. Downstream computational and experimental measurement locations were chosen to coincide to provide a proper comparison.

An existing annular turbine cascade was modified to provide a capability to measure blade midspan surface pressures. Experimental measurements of blade midspan surface pressures were obtained and favorably compared with numerical predictions over a range of subsonic and transonic conditions. Measurements were repeatable with a maximum repeatability error of less than 0.87%. Numerical solutions converged on pressure within approximately 1,200 iterations.

The minimum and maximum pressure values on the suction surface close to the trailing edge should be investigated further to confirm the influence of the relatively blunt trailing edge. An inlet boundary layer survey, requiring a rig modification, should be performed to supplement data in Reference 6 and refine the inlet boundary conditions to the computational model.

To supplement blade midspan surface pressures, blade pressure taps could be installed at other spanwise locations to include the hub, tip, and possibly endwall locations. A tip gap could be created and a computational tip gap model introduced to investigate the endwall region. Blade tips, and the adjacent endwall, could be instrumented for pressure readings and a numerical and experimental comparison conducted to obtain insight into tip leakage flows. In addition, an LDV optical-access window could be installed over the tip region to measure tip-gap flow characteristics.

Successful measurements of two-dimensional velocity, flow angle, and turbulence intensity through a 1.0922 mm diameter casing-access hole were demonstrated. All LDV data were acquired one-half axial chord downstream over seven radial positions and 17 separate peripheral displacements. Experimental flow velocities were increased from those of Reference 6 mainly due to the availability and use of a digital burst correlator in place of a signal processor. Although all LDV data reported herein were acquired at 0.9054 pressure ratio, measurements were attempted at 0.80 pressure ratio and some data were obtained; however, low data rate and endwall hole vibrations prevented a complete survey.



The comparison of computational and experimental Mach number and flow angle yielded an average 12.7% difference in Mach number and 3.2% (2.3 degrees) difference in flow angle. All experimental measurements were repeatable to an average uncertainty ranging from 0.2% to 7.0% for velocity, flow angle, and turbulence intensity in both random and coincidence data processor modes. All numerical and experimental comparisons were based on a circumferential match of maximum Mach number at the 80.9% (3.34 mm) span location. An investigation of other circumferential matching options should be conducted to explore the effect on the degree of agreement (i. e. minimum Mach number circumferential matching).

LDV alignment and radial survey procedures were devised and repeated. The probe volume passed cleanly through the optical access hole and radial surveys were conducted to provide complete coverage for data recording. Probe volume dimensions were further refined to aid in estimates of radial-position accuracy. Introduction of a second fiber-optics probe for three component measurements would decrease the effective probe volume and increase radial position accuracy. The ability to resolve endwall flow characteristics and eventually flow tip losses appears to be promising.

The present hardware did not provide pressure equalization across the optical access hole. A rig modification is required to equalize the pressure at the measurement location and eliminate any influence the opening might have on the flow characteristics. Also, the coarse peripheral (wake) positioning mechanism needs modification to allow more precise adjustment. Experimental peripheral positioning uncertainties may contribute to differences in the compared data.

Future efforts with computational fluid dynamics should include modeling the inlet bellmouth and center body, and conducting a turbulence model sensitivity analysis in the endwall region. Increased grid resolution in the endwall region would be beneficial. The experimental 99.9% span location coincided with the final k grid point and may account for the 70.8% difference between the experimental and computed Mach numbers. Finally, a detailed investigation into transonic and sonic flow conditions could be conducted to examine the blade-passage and trailing-edge shock locations and subsequent interactions, both computationally and experimentally.

## APPENDIX A. PRESSURE DATA ACQUISITION

Pressure data acquisition was accomplished using a Hewlett-Packard 9000 computer system with the following program:

```

10 !FILE NAME: BILLSP
20 !DISK LABEL: "/AUSTIN"
30 !LAST MODIFIED 10/27/94 BY LT BILL DONOVAN
40 !THIS PROGRAM RECORDS AND REDUCES MEASURED PRESSURES FROM A
50 !SCANIVALVE CONNECTED TO 11 STATIC PRESSURE PORTS ON ATC
60 !   VARIABLES / PORT CONNECTIONS...
70 !   V=DESIRED S/V
80 !   A=PRESENT S/V PORT
90 !   C=CHANNEL
100 !   Voltage ( )=DVM VOLTAGE READING FOR EACH PORT
110 !   S=SCANNER NUMBER
120 !   Pinhg=BAROMETRIC PRESSURE IN INHG
130 !   Pamb=BAROMETRIC PRESSURE (psi)
140 !   AMBIENT PRESSURE...#1
150 !   CALIBRATION PRESSURE...#2
160 !   REFERENCE UPSTREAM TOTAL PRESSURE...PORT #4
170 !   INNER HUB STATIC PRESSURE...PORT #5
180 !   UPSTREAM STATIC PRESSURE...PORT #6
190 !   ATC PASSAGE PORTS #1-#11...S/V PORTS #9-#19
200 !   Prat= INNER HUB STATIC/REFERENCE UPSTREAM TOTAL
210 !   Tstag=UPSTREAM STAGNATION TEMPERATURE IN DEGREES F
220 OPTION BASE 1
230 CLEAR SCREEN
240 PRINTER IS CRT
250 DISP "PLEASE WAIT WHILE RESETING SCANIVALVE TO PORT #1"
260 PRINT
270 V=5
280 A=1
290 GOSUB Read
300 DIM Y(20)
310 DIM Voltage(50)
320 DIM P(50)
330 INPUT "ENTER MONTH, DAY, YEAR (I.E.02, 04, 94)", Y(3), Y(5), Y(7)
340 INPUT "ENTER RUN #: ", Y(9)
350 PRINTER IS 711
360 PRINT USING "K, DD, " / " ", DD, " " / " ", DD", "DATE OF RUN: ", Y(3), Y(5), Y(7)
370 PRINT "DATA RUN ", Y(9)
380 PRINT USING "/", 5X, K, DDD, /, 2X, K, 15X, K, 17X, K", "SCANIVALVE #", V,
    "Port", "Volts", "Psi"
390 PRINTER IS CRT
400 PRINT "      ZERO DVM ON SCANIVALVE #5, PORT #1..."
410 PRINT
420 PRINT "      SET 20 inhg FOR SCANIVALVE CALIBRATION..."
430 PRINT
440 PRINT "      ADJUST SPAN FOR DVM = 0.009823 VOLTS ON PORT #2..."
450 PRINT
460 PRINT "ENSURE DESIRED PRESSURE RATIO IS SET..."
470 INPUT "ENTER BAROMETRIC PRESSURE IN INHG", Pinhg
480 INPUT "ENTER HIGHEST SCANIVALVE PORT # NEEDED ", Nports
490 INPUT "ENTER UPSTREAM STAGNATION TEMP. IN DEG. F ", Tstag

```

```

500      Pamb=Pinhg*(0.4911541)
510      PRINT "**** PRESS CONTINUE WHEN READY TO TAKE DATA****"
520      PAUSE
530      !*****
540      !RECORD THE PRESSURE DATA FOR THIS RUN
550      OUTPUT 722; "FIRIT1M0Z1"      ! SETS UP HP3456A DVM
560      V=5                          !SCANIVALVE #5 DESIGNATED
570      S=1                          !HP3495A SCAN #1 DESIGNATED
580      FOR A=1 TO Nports
590          GOSUB Read
600          WAIT 2.0
610          OUTPUT 701; "C"          !CLEARS SCANNER #1
620          OUTPUT 701 USING "DDD"; V+9 !SCAN SET TO READ S/V
630          TRIGGER 722
640          ENTER 722; Voltage(A)
650          P(A)=(Voltage(A) - Voltage(1))*1000+Pamb
660      PRINTER IS CRT
670      PRINT USING "2X,DDD,7X,7DD.DDDDDDD,8X,7DD.DDDD"; A, Voltage(A), P(A)
680      PRINTER IS 711
690      PRINT USING "2X,DDD,7X,7DD.DDDDDDD,8X,7DD.DDDD"; A, Voltage(A), P(A)
700      NEXT A
710      Prats=P(5)/P(4)
720      PRINT "Prat = ", Prats
730      PRINT "UPSTREAM STAGNATION TEMPERATURE WAS (DEG F)", Tstag
740      PRINT "INPUT BAROMETRIC PRESSURE IN (inhg) WAS ", Pinhg
750      !*****SUBROUTINE TO POSITION AND READ S/V PORTS*****
760      Read: OUTPUT 707 USING "#,K";V
770      P0=SPOOL(707)
780      L=BINAND(P0,15)
790      T1=ROTATE(P0,4)
800      M1=BINAND(T1,7)
810      P(A)=10*M1+L
820      CLEAR 707
830      IF P(A)=A THEN Finish
840      OUTPUT 701; "C"
850      OUTPUT 701 USING "DDD";V-1
860      OUTPUT 701; "C"
870      WAIT 0.1
880      GOTO Read
890      Finish: RETURN
900      END

```

The following table relates each Scanivalve port to its respective pressure measurement.

Scanivalve Port Number	Pressure Measured
1	Ambient Pressure
2	Calibration Pressure (set at 20 inches hg)
3	Not Used
4	Upstream Total Pressure ( $P_0$ )
5	Downstream Hub Static Pressure ( $P_{hub}$ )
6	Upstream Static Pressure ( $P_{static}$ )
7	Not Used
8	Not Used
9	Blade Static Port #1 (at leading edge)
10	Blade Static Port #2 (suction side)
11	Blade Static Port #3 (suction side)
12	Blade Static Port #4 (suction side)
13	Blade Static Port #5 (suction side)
14	Blade Static Port #6 (suction side)
15	Blade Static Port #7 (suction side)
16	Blade Static Port #8 (at leading edge)
17	Blade Static Port #9 (pressure side)
18	Blade Static Port #10 (pressure side)
19	Blade Static Port #11 (pressure side)

Table A1. Pressure Data Acquisition Connections



## APPENDIX B. GRID GENERATION INPUT FILE

Grid generation was completed on Silicon Graphics Indigo II workstations. The following TCGRID namelist file was utilized to obtain the final grid.

```
&nam1 im=150 jm=31 km=65 ijl=20 icap=18 k2d=3 merid=0 &end
&nam2 nle=16 nte=10 dsle=.018 dste=.003 dshub=.0004 dstip=.0004
      dswte=.001 dswex=.060 dstrh=1. dsmin=.0004 dsmax=.025
      dsra=.45 rcom=.098 &end
&nam3 item=100 idbg=0 0 0 0 0 0 0 aabb=.5 cadd=.45 &end
&nam4 zbc=-1.5 -1.5 0.5 -1.5 -1.5 .5
      rbc= 3.895 3.895 3.895 4.585 4.585 4.585 &end
'TRANSONIC TURBINE'
  2      2
  -.975 0.0
  3.895 3.895
  -.975 0.0
  4.585 4.585
  2      51      31
-7.8999996E-03 -9.9200001E-03 -1.2000000E-02 -1.4080000E-02
-1.6100001E-02 -1.7999999E-02 -1.9710001E-02 -2.1190001E-02
-2.2390001E-02 -2.3280000E-02 -8.1040002E-02 -0.1503800
-0.2308900 -0.3219400 -0.4228500 -0.5328500 -0.6511000
-0.7893000 -0.9275000 -0.9410600 -0.9527700 -0.9622700
-0.9692700 -0.9735600 -0.9750000 -0.9735600 -0.9692700
-0.9622700 -0.9527700 -0.9410700 -0.9275000 -0.9124900
-0.8250000 -0.7200000 -0.6000000 -0.4560000 -0.3440000
-0.2730000 -0.2200000 -0.1820000 -0.1160000 -5.9000000E-02
-1.7000001E-02 0.0000000E+00 -1.8220000E-04 -7.2359998E-04
-1.6100000E-03 -2.8100000E-03 -4.2900001E-03 -6.0000001E-03
-7.8999996E-03 -0.2349885 -0.2351271 -0.2351733 -0.2351271
-0.2349885 -0.2347600 -0.2344519 -0.2340719 -0.2336329
-0.2331451 -0.1980565 -0.1643980 -0.1324210 -0.1023671
-7.4464694E-02 -4.8926830E-02 -2.5946086E-02 -4.6726577E-03
1.6618744E-02 1.9057767E-02 2.2061616E-02 2.5545571E-02
2.9399229E-02 3.3504494E-02 3.7740692E-02 4.1976891E-02
4.6082158E-02 4.9935814E-02 5.3417202E-02 5.6423619E-02
5.8862645E-02 6.0659818E-02 6.9319643E-02 7.1887039E-02
6.7522466E-02 5.1347882E-02 2.5673941E-02 0.0000000E+00
-2.5673941E-02 -5.1347882E-02 -0.1026958 -0.1540437
-0.2053915 -0.2320924 -0.2326265 -0.2331451 -0.2336329
-0.2340719 -0.2344519 -0.2347600 -0.2349885 3.895000
3.895000 3.895000 3.895000 3.895000 3.895000 3.895000
3.895000 3.895000 3.895000 3.895000 3.895000 3.895000
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3.895000 3.895000 3.895000 3.895000 3.895000 3.895000
3.895000 3.895000 -7.8999996E-03 -9.9200001E-03
-1.2000000E-02 -1.4080000E-02 -1.6100001E-02 -1.7999999E-02
-1.9710001E-02 -2.1190001E-02 -2.2390001E-02 -2.3280000E-02
```

-8.1040002E-02 -0.1503800 -0.2308900 -0.3219400 -0.4228500  
 -0.5328500 -0.6511000 -0.7893000 -0.9275000 -0.9410600  
 -0.9527700 -0.9622700 -0.9692700 -0.9735600 -0.9750000  
 -0.9735600 -0.9692700 -0.9622700 -0.9527700 -0.9410700  
 -0.9275000 -0.9124900 -0.8250000 -0.7200000 -0.6000000  
 -0.4560000 -0.3440000 -0.2730000 -0.2200000 -0.1820000  
 -0.1160000 -5.9000000E-02 -1.7000001E-02 0.0000000E+00  
 -1.8220000E-04 -7.2359998E-04 -1.6100000E-03 -2.8100000E-03  
 -4.2900001E-03 -6.0000001E-03 -7.8999996E-03 -0.1996249  
 -0.1997426 -0.1997819 -0.1997426 -0.1996249 -0.1994308  
 -0.1991690 -0.1988462 -0.1984733 -0.1980589 -0.1682508  
 -0.1396576 -0.1124929 -8.6961828E-02 -6.3258447E-02  
 -4.1563794E-02 -2.2041440E-02 -3.9694658E-03 1.4117776E-02  
 1.6189748E-02 1.8741548E-02 2.1701200E-02 2.4974918E-02  
 2.8462378E-02 3.2061070E-02 3.5659760E-02 3.9147221E-02  
 4.2420939E-02 4.5378406E-02 4.7932386E-02 5.0004359E-02  
 5.1531076E-02 5.8887679E-02 6.1068702E-02 5.7360962E-02  
 4.3620501E-02 2.1810250E-02 0.0000000E+00 -2.1810250E-02  
 -4.3620501E-02 -8.7241001E-02 -0.1308615 -0.1744820  
 -0.1971647 -0.1976183 -0.1980589 -0.1984733 -0.1988462  
 -0.1991690 -0.1994308 -0.1996249 4.585000 4.585000 4.585000  
 4.585000 4.585000 4.585000 4.585000 4.585000 4.585000  
 4.585000 4.585000 4.585000 4.585000 4.585000 4.585000  
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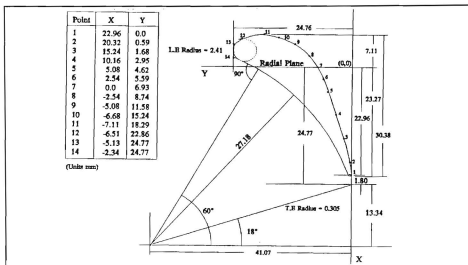


Figure B1. Blade Geometry From Ref. [6]

## APPENDIX C. RVC3D SAMPLE INPUT AND EXECUTION

The following input namelist example was for a 0.6815 pressure ratio with subsonic exit boundary conditions and Cebeci-Smith turbulence modeling.

```
TRANSONIC TURBINE Annular Cascade'
&nl1  im=150 jm=31 km=65 itl=20 iil=67 &end
&nl2  cfl=5.5 avisc1=0.0 avisc2=0.0 avisc4=0.50 ivdt=1 nstg=4 itmax=1200
&nl3  irs=1 epi=0.50 epj=0.60 epk=0.60 &end
&nl3  ibcin=3 ibcex=3 isymt=0 ires=10 icmt=50
&nl3  iresti=0 iresto=1 ibcpw=0 iqin=0 &end
&nl4  emxx=0.13 emty=0.0 emrz=0.0 expt=0.0 prat=0.6815 ga=1.4
&nl4  om=0.000000 igeom=1 alex=-67.0 &end
&nl5  ilt=3 tw=1.00 renr=6.651e6 pmr=.7 prtr=.9 vispwr=.666666
&nl5  srtp=0.0 cmutm=14. jedge=31 kedge=20 iltin=2 dblh=0.0048 dblt=0.0178 &end
&nl6  io1=1 io2=165 oar=0. ixjb=0 njo=1 nko=3
&nl6  jo=1 ko=5 11 16 &end
```

Runs on the NPS Cray computer took approximately 12 hours for a 1,200 iteration convergence and were accomplished with the command:

```
>qsub myjob.nqs
```

where the file myjob.nqs was as follows:

```
myjob.nqs
```

```
#QSUB -q prem -IT 50000 -lM 9Mw
cd /d1/whdonova
./exec.rvc3d < rvc3d.in > rvc3d.out
```

Runs on the NASA Ames Research Center Cray computer took approximately 1 hour for a 1,200 iteration convergence and were accomplished with the command:

```
>qsub -lM 10Mw -lt 14400 rvc3d.com
```

where the file rvc3d.com was as follows:

```
rvc3d.com
```

```
cd /m/vn/v3/wdonovan
./exec.rvc3d < rvc3d.in > rvc3d.out
```





# APPENDIX D. BLADE MIDSPAN SURFACE PRESSURE DATA

Prat = 0.6837		Date: 10/27/94, 1250 local, Run #1	Baro = 30.08 inHg	Tstag = 164°F
Scan/Valve Port #	Scan/Valve Port Value	Static Pressure (psf)	P/Po	
1	P ambient	14.7739		
2	P calibration	24.5716		
4	Po (Ref. Upstream Stagnation)	22.2306	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.1982		
6	P up static (Ref. Upstream Static)	22.2190		
9	P Static Port 1 (@ leading edge)	22.2524	1.0010	
10	P Static Port 2	21.3661	0.9611	
11	P Static Port 3	19.3768	0.8806	
12	P Static Port 4	16.9327	0.7617	
13	P Static Port 5	15.4186	0.6936	
14	P Static Port 6	15.6884	0.7027	
15	P Static Port 7	15.4516	0.6951	
16	P Static Port 8 (@ leading edge)	22.2242	0.9997	
17	P Static Port 9	22.0043	0.9898	
18	P Static Port 10	21.0074	0.9450	
19	P Static Port 11	20.0537	0.9021	
Prat = 0.6828		Date: 10/27/94, 1245 local, Run #2	Baro = 30.08 inHg	Tstag = 165°F
Scan/Valve Port #	Scan/Valve Port Value	Static Pressure (psf)	P/Po	
1	P ambient	14.7739		
2	P calibration	24.5786		
4	Po (Ref. Upstream Stagnation)	22.2571	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.1969		
6	P up static (Ref. Upstream Static)	22.2622		
9	P Static Port 1 (@ leading edge)	22.2576	1.0000	
10	P Static Port 2	21.3866	0.9609	
11	P Static Port 3	19.5839	0.8799	
12	P Static Port 4	16.9291	0.7606	
13	P Static Port 5	15.3810	0.6911	
14	P Static Port 6	15.6660	0.7039	
15	P Static Port 7	15.4305	0.6933	
16	P Static Port 8 (@ leading edge)	22.2670	1.0004	
17	P Static Port 9	22.0659	0.9914	
18	P Static Port 10	21.0580	0.9461	
19	P Static Port 11	20.1035	0.9032	
Prat = 0.6820		Date: 10/27/94, 1255 local, Run #3	Baro = 30.09 inHg	Tstag = 166°F
Scan/Valve Port #	Scan/Valve Port Value	Static Pressure (psf)	P/Po	
1	P ambient	14.7813		
2	P calibration	24.5609		
4	Po (Ref. Upstream Stagnation)	22.2541	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.1779		
6	P up static (Ref. Upstream Static)	22.2372		
9	P Static Port 1 (@ leading edge)	22.2562	1.0001	
10	P Static Port 2	21.3422	0.9590	
11	P Static Port 3	19.5633	0.8791	
12	P Static Port 4	16.9163	0.7601	
13	P Static Port 5	15.4302	0.6934	
14	P Static Port 6	15.6781	0.7045	
15	P Static Port 7	15.4566	0.6946	
16	P Static Port 8 (@ leading edge)	22.2340	0.9991	
17	P Static Port 9	22.0180	0.9804	
18	P Static Port 10	21.0070	0.9440	
19	P Static Port 11	20.0738	0.9020	

Table D1. 0.6815 Pressure Ratio Series #1

Prat = 0.6779		Date: 11/02/94, 1436 local, Run #4	Baro = 29.84 inHg	Tstag = 98°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.6560		
2	P calibration	24.4824		
4	Po (Ref. Upstream Stagnation)	22.2493	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.0818		
6	P up static (Ref. Upstream Static)	22.2322		
9	P Static Port 1 (@ leading edge)	22.2633	1.0006	
10	P Static Port 2	21.3571	0.9599	
11	P Static Port 3	19.5428	0.8784	
12	P Static Port 4	16.8398	0.7569	
13	P Static Port 5	15.3090	0.6881	
14	P Static Port 6	15.5667	0.6996	
15	P Static Port 7	15.3472	0.6898	
16	P Static Port 8 (@ leading edge)	22.2201	0.9987	
17	P Static Port 9	21.9913	0.9884	
18	P Static Port 10	20.9822	0.9430	
19	P Static Port 11	20.0285	0.9002	
Prat = 0.6809		Date: 11/02/94, 1445 local, Run #5	Baro = 29.84 inHg	Tstag = 101°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.6560		
2	P calibration	24.4829		
4	Po (Ref. Upstream Stagnation)	22.1573	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.0879		
6	P up static (Ref. Upstream Static)	22.1395		
9	P Static Port 1 (@ leading edge)	22.1270	0.9986	
10	P Static Port 2	21.2561	0.9593	
11	P Static Port 3	19.4558	0.8781	
12	P Static Port 4	16.8161	0.7589	
13	P Static Port 5	15.2810	0.6897	
14	P Static Port 6	15.5741	0.7029	
15	P Static Port 7	15.3099	0.6910	
16	P Static Port 8 (@ leading edge)	22.1201	0.9983	
17	P Static Port 9	21.9032	0.9885	
18	P Static Port 10	20.8871	0.9427	
19	P Static Port 11	19.9552	0.9006	
Prat = 0.6861		Date: 11/02/94, 1455 local, Run #6	Baro = 29.84 inHg	Tstag = 102°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.6560		
2	P calibration	24.4737		
4	Po (Ref. Upstream Stagnation)	22.1854	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.0893		
6	P up static (Ref. Upstream Static)	22.1707		
9	P Static Port 1 (@ leading edge)	22.1671	0.9992	
10	P Static Port 2	21.2940	0.9598	
11	P Static Port 3	19.4914	0.8786	
12	P Static Port 4	16.8298	0.7586	
13	P Static Port 5	15.3149	0.6903	
14	P Static Port 6	15.5797	0.7023	
15	P Static Port 7	15.3721	0.6929	
16	P Static Port 8 (@ leading edge)	22.1570	0.9987	
17	P Static Port 9	21.9304	0.9885	
18	P Static Port 10	20.9124	0.9426	
19	P Static Port 11	19.9889	0.9010	

Table D2. 0.6815 Pressure Ratio Series #2

Prat = 0.5060		Date: 11/21/94, 1600 local, Run #1	Baro = 30.09 inHg	Tstag = 115°F
Scan Valve Port #	Scan Valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.6074		
4	Po (Ref. Upstream Stagnation)	30.5239	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.4450		
6	P up static (Ref. Upstream Static)	30.8693		
9	P Static Port 1 (@ leading edge)	30.8956	1.0122	
10	P Static Port 2	29.4056	0.9634	
11	P Static Port 3	26.3156	0.8621	
12	P Static Port 4	21.3339	0.6989	
13	P Static Port 5	16.5247	0.5414	
14	P Static Port 6	17.0462	0.5585	
15	P Static Port 7	15.8206	0.5183	
16	P Static Port 8 (@ leading edge)	30.8472	1.0106	
17	P Static Port 9	30.4413	0.9973	
18	P Static Port 10	28.7061	0.9404	
19	P Static Port 11	27.0593	0.8865	
Prat = 0.5080		Date: 11/21/94, 1605 local, Run #2	Baro = 30.09 inHg	Tstag = 115°F
Scan Valve Port #	Scan Valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.6073		
4	Po (Ref. Upstream Stagnation)	30.5541	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.5215		
6	P up static (Ref. Upstream Static)	30.8504		
9	P Static Port 1 (@ leading edge)	30.9060	1.0115	
10	P Static Port 2	29.4090	0.9625	
11	P Static Port 3	26.3010	0.8608	
12	P Static Port 4	21.3622	0.6992	
13	P Static Port 5	16.6027	0.5434	
14	P Static Port 6	17.0540	0.5582	
15	P Static Port 7	15.9400	0.5217	
16	P Static Port 8 (@ leading edge)	30.8135	1.0085	
17	P Static Port 9	30.4450	0.9964	
18	P Static Port 10	28.7116	0.9397	
19	P Static Port 11	27.0684	0.8859	
Prat = 0.5070		Date: 11/21/94, 1610 local, Run #3	Baro = 30.09 inHg	Tstag = 115°F
Scan Valve Port #	Scan Valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.6079		
4	Po (Ref. Upstream Stagnation)	30.6047	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.5163		
6	P up static (Ref. Upstream Static)	30.9081		
9	P Static Port 1 (@ leading edge)	30.9457	1.0111	
10	P Static Port 2	29.4551	0.9624	
11	P Static Port 3	26.3531	0.8611	
12	P Static Port 4	21.3499	0.6976	
13	P Static Port 5	16.5516	0.5408	
14	P Static Port 6	17.0929	0.5585	
15	P Static Port 7	15.9170	0.5201	
16	P Static Port 8 (@ leading edge)	30.7752	1.0056	
17	P Static Port 9	30.3840	0.9928	
18	P Static Port 10	28.6649	0.9366	
19	P Static Port 11	27.0382	0.8835	

Table D3. 0.5070 Pressure Ratio Series

Prat = 0.6050		Date: 11/21/94, 1535 local, Run #1	Baro = 30.09 inHg	Tstag = 102°F
Scanivalve Port #	Scanivalve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.6010		
4	Po (Ref. Upstream Stagnation)	25.3685	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.3474		
6	P up static (Ref. Upstream Static)	25.3969		
9	P Static Port 1 (@ leading edge)	25.3960	1.0011	
10	P Static Port 2	24.2663	0.9566	
11	P Static Port 3	21.9130	0.8638	
12	P Static Port 4	18.3058	0.7216	
13	P Static Port 5	15.7541	0.6210	
14	P Static Port 6	16.1405	0.6362	
15	P Static Port 7	15.6129	0.6154	
16	P Static Port 8 (@ leading edge)	25.3701	1.0001	
17	P Static Port 9	25.0731	0.9884	
18	P Static Port 10	23.7612	0.9366	
19	P Static Port 11	22.5170	0.8876	
Prat = 0.6045		Date: 11/21/94, 1545 local, Run #2	Baro = 30.09 inHg	Tstag = 105°F
Scanivalve Port #	Scanivalve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.5934		
4	Po (Ref. Upstream Stagnation)	25.3393	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.3187		
6	P up static (Ref. Upstream Static)	25.3541		
9	P Static Port 1 (@ leading edge)	25.4161	1.0030	
10	P Static Port 2	24.3015	0.9590	
11	P Static Port 3	21.9296	0.8654	
12	P Static Port 4	18.3235	0.7231	
13	P Static Port 5	15.7793	0.6227	
14	P Static Port 6	16.1046	0.6356	
15	P Static Port 7	15.6232	0.6166	
16	P Static Port 8 (@ leading edge)	25.2650	0.9971	
17	P Static Port 9	24.9943	0.9864	
18	P Static Port 10	23.6995	0.9353	
19	P Static Port 11	22.4807	0.8872	
Prat = 0.6026		Date: 11/21/94, 1555 local, Run #3	Baro = 30.09 inHg	Tstag = 105°F
Scanivalve Port #	Scanivalve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.5966		
4	Po (Ref. Upstream Stagnation)	25.3541	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.2796		
6	P up static (Ref. Upstream Static)	25.4145		
9	P Static Port 1 (@ leading edge)	25.3978	1.0017	
10	P Static Port 2	24.2691	0.9572	
11	P Static Port 3	21.9214	0.8646	
12	P Static Port 4	18.3097	0.7222	
13	P Static Port 5	15.7499	0.6212	
14	P Static Port 6	16.1173	0.6357	
15	P Static Port 7	15.6638	0.6178	
16	P Static Port 8 (@ leading edge)	25.3317	0.9991	
17	P Static Port 9	25.0652	0.9886	
18	P Static Port 10	23.7301	0.9367	
19	P Static Port 11	22.5026	0.8875	

Table D4. 0.6041 Pressure Ratio Series

Prat = 0.8683		Date: 11/21/94, 1505 locn, Run #1	Baro = 30.09 inHg	Tstag = 95°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.6032		
4	Po (Ref. Upstream Stagnation)	18.5613	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.0026		
6	P up static (Ref. Upstream Static)	18.5834		
9	P Static Port 1 (@ leading edge)	18.5658	1.0002	
10	P Static Port 2	18.0638	0.9732	
11	P Static Port 3	17.0140	0.9166	
12	P Static Port 4	15.6507	0.8432	
13	P Static Port 5	15.0818	0.8125	
14	P Static Port 6	15.2191	0.8199	
15	P Static Port 7	15.1762	0.8176	
16	P Static Port 8 (@ leading edge)	18.5516	0.9995	
17	P Static Port 9	18.4358	0.9932	
18	P Static Port 10	17.8276	0.9605	
19	P Static Port 11	17.3184	0.9330	
Prat = 0.8671		Date: 11/21/94, 1515 locn, Run #2	Baro = 30.09 inHg	Tstag = 96°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.6037		
4	Po (Ref. Upstream Stagnation)	18.5866	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.0020		
6	P up static (Ref. Upstream Static)	18.5665		
9	P Static Port 1 (@ leading edge)	18.5504	0.9981	
10	P Static Port 2	18.0527	0.9713	
11	P Static Port 3	17.0381	0.9167	
12	P Static Port 4	15.6733	0.8433	
13	P Static Port 5	15.0533	0.8099	
14	P Static Port 6	15.2127	0.8185	
15	P Static Port 7	15.1776	0.8166	
16	P Static Port 8 (@ leading edge)	18.5620	0.9987	
17	P Static Port 9	18.4420	0.9922	
18	P Static Port 10	17.8390	0.9598	
19	P Static Port 11	17.3165	0.9317	
Prat = 0.8678		Date: 11/21/94, 1525 locn, Run #3	Baro = 30.09 inHg	Tstag = 96°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.6012		
4	Po (Ref. Upstream Stagnation)	18.5811	1.0000	
5	P hub exit (Static @ 0.5c downstream)	15.0096		
6	P up static (Ref. Upstream Static)	18.5759		
9	P Static Port 1 (@ leading edge)	18.5878	1.0004	
10	P Static Port 2	18.1007	0.9741	
11	P Static Port 3	17.0542	0.9178	
12	P Static Port 4	15.6716	0.8434	
13	P Static Port 5	15.0769	0.8114	
14	P Static Port 6	15.2120	0.8187	
15	P Static Port 7	15.1707	0.8165	
16	P Static Port 8 (@ leading edge)	18.5753	0.9997	
17	P Static Port 9	18.4218	0.9914	
18	P Static Port 10	17.8355	0.9599	
19	P Static Port 11	17.3416	0.9333	

Table D5. 0.8077 Pressure Ratio Series

Prat = 0.9063		Date: 11/21/94, 1430 local, Run #1	Baro = 30.09 inHg	Tstag = 90°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.6056		
4	Po (Ref. Upstream Stagnation)	16.4287	1.0000	
5	P hub exit (Static @ 0.5c downstream)	14.8858		
6	P up static (Ref. Upstream Static)	16.4464		
9	P Static Port 1 (@ leading edge)	16.4600	1.0019	
10	P Static Port 2	16.2173	0.9871	
11	P Static Port 3	15.7200	0.9569	
12	P Static Port 4	15.1043	0.9194	
13	P Static Port 5	14.8879	0.9062	
14	P Static Port 6	14.9555	0.9103	
15	P Static Port 7	14.9519	0.9101	
16	P Static Port 8 (@ leading edge)	16.4225	0.9996	
17	P Static Port 9	16.4047	0.9985	
18	P Static Port 10	16.1224	0.9814	
19	P Static Port 11	15.8879	0.9671	
Prat = 0.9054		Date: 11/21/94, 1445 local, Run #2	Baro = 30.09 inHg	Tstag = 91°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7880		
2	P calibration	24.5959		
4	Po (Ref. Upstream Stagnation)	16.4376	1.0000	
5	P hub exit (Static @ 0.5c downstream)	14.8830		
6	P up static (Ref. Upstream Static)	16.4723		
9	P Static Port 1 (@ leading edge)	16.4365	0.9999	
10	P Static Port 2	16.2328	0.9875	
11	P Static Port 3	15.7218	0.9565	
12	P Static Port 4	15.1136	0.9195	
13	P Static Port 5	14.8797	0.9052	
14	P Static Port 6	14.9571	0.9099	
15	P Static Port 7	14.9630	0.9103	
16	P Static Port 8 (@ leading edge)	16.4432	1.0003	
17	P Static Port 9	16.3780	0.9964	
18	P Static Port 10	16.1170	0.9805	
19	P Static Port 11	15.8913	0.9668	
Prat = 0.9046		Date: 11/21/94, 1455 local, Run #3	Baro = 30.09 inHg	Tstag = 93°F
Scan/valve Port #	Scan/valve Port Value	Static Pressure (psi)	P/Po	
1	P ambient	14.7788		
2	P calibration	24.5980		
4	Po (Ref. Upstream Stagnation)	16.4384	1.0000	
5	P hub exit (Static @ 0.5c downstream)	14.8704		
6	P up static (Ref. Upstream Static)	16.4480		
9	P Static Port 1 (@ leading edge)	16.4672	1.0018	
10	P Static Port 2	16.2248	0.9870	
11	P Static Port 3	15.7233	0.9565	
12	P Static Port 4	15.1201	0.9198	
13	P Static Port 5	14.8891	0.9058	
14	P Static Port 6	14.9702	0.9107	
15	P Static Port 7	14.9636	0.9103	
16	P Static Port 8 (@ leading edge)	16.4591	1.0013	
17	P Static Port 9	16.3869	0.9969	
18	P Static Port 10	16.1301	0.9812	
19	P Static Port 11	15.8869	0.9665	

Table D6. 0.9054 Pressure Ratio Series

# APPENDIX E. LDV DATA

Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	95.0 30.03
-8	0.01	83.2	21.8	86.0	0.244	75.3	7.76	5.19	P stag (inHg)	3.13
-8	0.06	85.7	21.9	88.3	0.252	75.7	8.37	5.26	Prat	0.9056
-8	0.18	87.1	22.5	90.0	0.256	75.5	7.83	5.06	M exit	0.379
-8	0.42	92.5	22.6	95.2	0.271	76.3	7.60	5.08	T exit (°K)	299.4
-8	0.89	97.3	22.7	99.9	0.284	76.9	7.12	4.23	a exit (m/s)	346.84
-8	1.78	99.0	22.8	101.6	0.289	77.0	5.41	3.98	a stag (m/s)	351.79
-8	3.34	102.0	24.6	104.9	0.298	76.4	5.39	3.98	V exit (m/s)	131.46
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	95.0 30.03
-7	0.01	86.8	21.4	89.4	0.254	76.2	7.56	5.40	P stag (inHg)	3.13
-7	0.06	87.3	21.8	90.0	0.256	76.0	8.29	5.52	Prat	0.9056
-7	0.18	89.4	22.3	92.1	0.262	76.0	7.83	5.63	M exit	0.379
-7	0.42	92.0	22.6	94.7	0.269	76.2	7.69	5.17	T exit (°K)	299.4
-7	0.89	96.6	23.0	99.3	0.282	76.6	6.96	4.85	a exit (m/s)	346.84
-7	1.78	98.4	23.0	101.1	0.287	76.8	5.49	4.18	a stag (m/s)	351.79
-7	3.34	100.0	24.5	103.0	0.293	76.2	5.09	4.59	V exit (m/s)	131.46
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	95.0 30.03
-6	0.01	86.1	21.8	88.8	0.252	75.8	7.09	5.92	P stag (inHg)	3.13
-6	0.06	87.6	21.9	90.3	0.257	76.0	7.56	5.66	Prat	0.9056
-6	0.18	88.4	22.6	91.2	0.259	75.7	8.29	5.84	M exit	0.379
-6	0.42	91.7	23.3	94.6	0.269	75.7	7.58	5.71	T exit (°K)	299.4
-6	0.89	96.2	23.0	98.9	0.281	76.6	6.40	5.19	a exit (m/s)	346.84
-6	1.78	99.7	23.5	102.4	0.291	76.7	5.26	4.29	a stag (m/s)	351.79
-6	3.34	103.0	24.4	105.9	0.301	76.7	6.68	4.11	V exit (m/s)	131.46
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	96.0 30.03
-5	0.01	85.6	21.8	88.3	0.251	75.7	7.60	6.32	P stag (inHg)	3.13
-5	0.06	86.9	22.3	89.7	0.255	75.6	8.13	6.39	Prat	0.9056
-5	0.18	89.7	22.4	92.5	0.263	76.0	7.29	5.90	M exit	0.379
-5	0.42	91.9	22.8	94.7	0.269	76.1	7.34	5.95	T exit (°K)	299.9
-5	0.89	95.8	23.6	98.7	0.280	76.2	6.81	5.43	a exit (m/s)	347.15
-5	1.78	98.6	23.7	101.4	0.288	76.3	5.11	4.75	a stag (m/s)	352.10
-5	3.34	102.0	24.6	104.9	0.298	76.4	5.97	4.13	V exit (m/s)	131.58

Table E1. LDV Data (Wake Positions of -8, -7, -6, and -5 Degrees)



Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	96.0 30.03
-4	0.01	86.1	22.8	89.1	0.253	75.2	7.35	6.45	P stag (inHg)	3.13
-4	0.06	87.3	22.4	90.1	0.256	75.6	7.04	6.50	Prat	0.9056
-4	0.18	88.5	24.1	91.7	0.260	74.8	7.56	6.25	M exit	0.379
-4	0.42	90.9	22.6	93.7	0.266	76.0	6.95	5.93	T exit (°K)	299.9
-4	0.89	96.6	24.6	99.7	0.283	75.7	6.78	5.44	a exit (m/s)	347.15
-4	1.78	98.9	24.4	101.9	0.289	76.1	4.42	4.63	a stag (m/s)	352.10
-4	3.34	103.0	25.7	106.2	0.302	76.0	5.66	3.87	V exit (m/s)	131.58
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	96.0 30.03
-3	0.01	85.4	23.4	88.5	0.251	74.7	7.34	6.57	P stag (inHg)	3.13
-3	0.06	86.4	22.8	89.4	0.254	75.2	7.26	6.11	Prat	0.9056
-3	0.18	87.9	22.5	90.7	0.258	75.6	6.74	6.33	M exit	0.379
-3	0.42	90.8	24.1	93.9	0.267	75.1	6.77	5.97	T exit (°K)	299.9
-3	0.89	95.0	24.4	98.1	0.279	75.6	6.27	5.36	a exit (m/s)	347.15
-3	1.78	100.0	24.6	103.0	0.293	76.2	4.30	4.68	a stag (m/s)	352.10
-3	3.34	103.0	26.1	106.3	0.302	75.8	3.91	3.62	V exit (m/s)	131.58
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	96.0 30.03
-2	0.01	87.7	23.7	90.8	0.258	74.9	7.83	6.19	P stag (inHg)	3.13
-2	0.06	87.4	23.8	90.6	0.257	74.8	7.98	5.92	Prat	0.9056
-2	0.18	89.2	24.1	92.4	0.262	74.9	7.75	6.04	M exit	0.379
-2	0.42	92.1	24.4	95.3	0.271	75.2	7.83	5.92	T exit (°K)	299.9
-2	0.89	94.9	24.3	98.0	0.278	75.6	6.68	5.44	a exit (m/s)	347.15
-2	1.78	100.0	24.4	102.9	0.292	76.3	4.34	4.92	a stag (m/s)	352.10
-2	3.34	104.0	25.6	107.1	0.304	76.2	3.73	4.03	V exit (m/s)	131.58
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	96.0 30.03
-1	0.01	88.5	24.0	91.7	0.260	74.8	8.36	5.66	P stag (inHg)	3.13
-1	0.06	89.0	23.5	92.1	0.262	75.2	7.42	5.40	Prat	0.9056
-1	0.18	90.4	23.9	93.5	0.266	75.2	7.75	5.54	M exit	0.379
-1	0.42	93.0	23.6	95.9	0.272	75.8	7.00	5.46	T exit (°K)	299.9
-1	0.89	97.0	24.5	100.0	0.284	75.8	6.37	4.96	a exit (m/s)	347.15
-1	1.78	99.6	24.8	102.6	0.291	76.0	4.72	4.05	a stag (m/s)	352.10
-1	3.34	105.0	26.3	108.2	0.307	75.9	3.75	3.93	V exit (m/s)	131.58

Table E2. LDV Data (Wake Positions of -4, -3, -2, and -1 Degrees)

Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F)	96.0
0	0.01	90.2	24.7	93.5	0.266	74.7	8.47	5.12	P static (inHg)	30.03
0	0.06	92.6	24.5	95.8	0.272	75.2	7.68	4.89	P stag (inHg)	3.13
0	0.18	92.6	24.5	95.8	0.272	75.2	7.64	4.68	Prat	0.9056
0	0.42	98.4	24.9	101.5	0.288	75.8	7.03	4.54	M exit	0.379
0	0.89	101.7	24.5	104.6	0.297	76.5	6.10	4.13	T exit (°K)	299.9
0	1.78	103.6	24.8	106.5	0.302	76.5	3.90	3.88	a exit (m/s)	347.15
0	3.34	106.8	27.1	110.2	0.313	75.7	2.87	3.23	a stag (m/s)	352.10
0									V exit (m/s)	131.58
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F)	100.0
1	0.01	89.1	24.4	92.4	0.261	74.7	8.18	4.78	P static (inHg)	30.03
1	0.06	91.1	24.3	94.3	0.267	75.1	7.73	4.63	P stag (inHg)	3.13
1	0.18	93.1	23.9	96.1	0.272	75.6	8.03	4.59	Prat	0.9056
1	0.42	96.9	24.3	99.9	0.283	75.9	7.24	4.11	M exit	0.379
1	0.89	100.0	24.2	102.9	0.291	76.4	5.97	3.77	T exit (°K)	302.1
1	1.78	104.0	24.2	106.8	0.302	76.9	3.96	3.68	a exit (m/s)	348.40
1	3.34	107.0	26.7	110.3	0.312	76.0	3.00	3.67	a stag (m/s)	353.37
1									V exit (m/s)	132.05
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F)	100.0
2	0.01	91.4	23.2	94.3	0.267	75.8	8.33	4.42	P static (inHg)	30.03
2	0.06	90.7	23.2	93.6	0.265	75.7	7.95	4.09	P stag (inHg)	3.13
2	0.18	93.0	22.8	95.8	0.271	76.2	8.18	4.45	Prat	0.9056
2	0.42	96.4	23.3	99.2	0.281	76.4	7.30	3.68	M exit	0.379
2	0.89	101.0	23.6	103.7	0.293	76.8	6.27	3.30	T exit (°K)	302.1
2	1.78	103.0	24.2	105.8	0.299	76.8	4.66	3.09	a exit (m/s)	348.40
2	3.34	106.0	26.3	109.2	0.309	76.1	3.95	3.66	a stag (m/s)	353.37
2									V exit (m/s)	132.05
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F)	98.0
3	0.01	89.3	22.5	92.1	0.261	75.9	8.42	4.74	P static (inHg)	30.03
3	0.06	90.4	22.3	93.1	0.264	76.1	7.89	4.64	P stag (inHg)	3.13
3	0.18	91.9	22.9	94.7	0.268	76.0	8.27	4.01	Prat	0.9056
3	0.42	95.1	22.3	97.7	0.277	76.8	7.66	3.94	M exit	0.379
3	0.89	98.2	22.8	100.8	0.286	76.9	7.06	3.62	T exit (°K)	301.0
3	1.78	102.0	23.2	104.6	0.297	77.2	5.28	3.18	a exit (m/s)	347.78
3	3.34	104.0	25.6	107.1	0.304	76.2	4.83	3.23	a stag (m/s)	352.74
3									V exit (m/s)	131.82

Table E3. LDV Data (Wake Positions of 0, 1, 2, and 3 Degrees)

Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inhg)	97.0 30.03
4	0.01	87.5	21.8	90.2	0.236	76.0	8.73	4.64	P stag (inhg)	3.13
4	0.06	88.1	22.2	90.9	0.258	75.9	8.12	4.53	Prat	0.9056
4	0.18	90.4	21.9	93.0	0.264	76.4	7.90	4.69	M exit	0.379
4	0.42	91.8	22.0	94.4	0.268	76.5	7.82	4.28	T exit (°K)	306.5
4	0.89	97.8	22.2	100.3	0.285	77.2	6.92	4.23	a exit (m/s)	347.46
4	0.89	101.0	22.9	103.6	0.294	77.2	6.08	3.82	a stag (m/s)	352.42
4	3.34	102.0	24.8	105.0	0.298	76.3	6.12	3.85	V exit (m/s)	131.70
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inhg)	96.0 30.03
5	0.01	85.2	22.7	88.2	0.250	75.1	7.28	5.66	P stag (inhg)	3.13
5	0.06	87.7	22.7	90.6	0.257	75.5	8.36	5.63	Prat	0.9056
5	0.18	89.4	22.6	92.2	0.262	75.8	7.60	5.65	M exit	0.379
5	0.42	92.9	22.8	95.7	0.272	76.2	7.83	5.25	T exit (°K)	299.9
5	0.89	97.4	23.3	100.1	0.284	76.5	6.93	5.12	a exit (m/s)	281.71
5	1.78	99.1	22.5	101.6	0.289	77.2	5.36	4.83	a stag (m/s)	285.73
5	3.34	99.8	24.2	102.7	0.292	76.4	5.24	4.07	V exit (m/s)	106.79
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inhg)	96.0 30.03
6	0.01	85.9	22.0	88.7	0.252	75.6	7.56	5.85	P stag (inhg)	3.13
6	0.06	85.1	22.8	88.1	0.250	75.0	7.13	6.05	Prat	0.9056
6	0.18	87.9	22.9	90.8	0.258	75.4	7.23	6.14	M exit	0.379
6	0.42	91.1	23.1	94.0	0.267	75.8	7.11	5.76	T exit (°K)	299.9
6	0.89	94.7	23.4	97.5	0.277	76.1	6.66	5.36	a exit (m/s)	347.13
6	1.78	99.5	23.7	102.3	0.291	76.6	5.12	4.71	a stag (m/s)	352.10
6	3.34	101.0	25.0	104.0	0.295	76.1	4.39	4.02	V exit (m/s)	131.58
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inhg)	96.0 30.03
7	0.01	83.9	22.9	87.0	0.247	74.7	7.39	6.52	P stag (inhg)	3.13
7	0.06	86.6	23.2	89.7	0.255	75.0	7.59	6.19	Prat	0.9056
7	0.18	86.8	23.0	89.8	0.255	75.2	7.68	6.10	M exit	0.379
7	0.42	89.5	23.3	92.5	0.263	75.4	6.53	6.07	T exit (°K)	299.9
7	0.89	94.9	24.2	97.9	0.278	75.7	6.79	5.39	a exit (m/s)	347.13
7	1.78	98.5	25.1	101.6	0.289	75.7	4.88	4.48	a stag (m/s)	352.10
7	3.34	102.0	24.9	105.0	0.298	76.3	4.24	4.80	V exit (m/s)	131.58

Table E4. LDV Data (Wake Positions of 4, 5, 6, and 7 Degrees)

Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F)	96.0
8	0.01	85.8	23.1	88.9	0.252	74.9	7.68	6.08	P stag (inHg)	3.03
8	0.06	86.7	23.2	89.8	0.255	75.0	8.21	6.19	Prat	0.9056
8	0.18	88.4	23.3	91.4	0.260	75.2	6.94	6.66	M exit	0.379
8	0.42	91.9	23.7	94.9	0.270	75.5	7.08	6.18	T exit (°K)	299.9
8	0.89	95.3	24.1	98.3	0.279	75.8	6.68	5.50	a exit (m/s)	347.15
8	1.78	100.0	24.7	103.0	0.293	76.1	4.57	4.65	a stag (m/s)	352.10
8	3.34	103.0	25.5	106.1	0.301	76.1	3.83	3.86	V exit (m/s)	131.58

Table E5. LDV Data (Wake Position of 8 Degrees)



# APPENDIX F. LDV REPEATABILITY DATA

Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	96.0 29.99
Coincidence -8	0.01	90.4	22.4	93.1	0.264	76.1	8.74	4.61	P stag (inHg)	3.13
Coincidence -8	0.06	91.0	22.6	93.8	0.266	76.1	8.74	4.60	Prat	0.9055
Coincidence -8	0.18	92.3	22.2	94.9	0.270	76.5	8.66	4.54	M exit	0.379
Coincidence -8	0.42	96.4	22.1	98.9	0.281	77.1	8.28	4.48	T exit (°K)	299.9
Coincidence -8	0.89	99.5	22.2	101.9	0.289	77.4	7.58	4.27	a exit (m/s)	347.15
Coincidence -8	1.78	102.0	22.6	104.5	0.297	77.5	5.91	3.95	a stag (m/s)	352.10
Coincidence -8	3.34	104.0	25.5	107.1	0.304	76.2	4.88	4.24	V exit (m/s)	131.58
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	96.0 30.00
Random -8	0.01	89.8	22.4	92.6	0.263	76.0	8.97	4.74	P stag (inHg)	3.13
Random -8	0.06	91.4	22.5	94.1	0.267	76.2	8.28	4.67	Prat	0.9055
Random -8	0.18	92.7	22.5	95.4	0.271	76.4	8.51	4.54	M exit	0.379
Random -8	0.42	95.6	21.7	98.0	0.278	77.2	8.74	4.44	T exit (°K)	299.9
Random -8	0.89	99.6	22.2	102.0	0.290	77.4	7.52	4.23	a exit (m/s)	347.15
Random -8	1.78	101.0	22.8	103.5	0.294	77.3	5.83	3.58	a stag (m/s)	352.10
Random -8	3.34	104.0	25.8	107.2	0.304	76.1	5.16	3.65	V exit (m/s)	131.58
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	93.0 30.00
Coincidence 0	0.01	90.6	24.3	93.8	0.267	75.0	8.46	4.73	P stag (inHg)	3.13
Coincidence 0	0.06	92.4	23.9	95.4	0.272	75.5	7.85	5.17	Prat	0.9055
Coincidence 0	0.18	94.0	24.0	97.0	0.276	75.7	8.23	5.14	M exit	0.379
Coincidence 0	0.42	96.9	24.0	99.8	0.284	76.1	7.32	4.59	T exit (°K)	298.3
Coincidence 0	0.89	100.0	23.8	102.8	0.293	76.6	6.58	4.23	a exit (m/s)	346.21
Coincidence 0	1.78	104.0	24.7	106.9	0.304	76.6	4.33	3.33	a stag (m/s)	351.15
Coincidence 0	3.34	107.0	27.1	110.4	0.314	75.8	3.39	3.28	V exit (m/s)	131.23
Wake Position (degrees)	Depth (mm)	V-theta (m/s)	V-z (m/s)	V total (m/s)	Mach Number	Flow Angle (degrees)	V-theta turb. int.	V-z turb. int.	T stag (°F) P static (inHg)	87.0 30.00
Random 0	0.01	91.5	22.9	94.3	0.270	75.9	8.43	5.27	P stag (inHg)	3.13
Random 0	0.06	92.9	23.8	95.9	0.275	75.6	8.05	4.83	Prat	0.9055
Random 0	0.18	94.4	23.9	97.4	0.279	75.8	7.74	4.64	M exit	0.379
Random 0	0.42	97.5	24.1	100.4	0.287	76.1	6.87	4.56	T exit (°K)	295.1
Random 0	0.89	101.0	24.1	103.8	0.297	76.6	5.79	3.82	a exit (m/s)	344.33
Random 0	1.78	104.0	25.0	107.0	0.306	76.5	4.32	3.18	a stag (m/s)	349.24
Random 0	3.34	107.0	26.6	110.3	0.316	76.0	3.10	3.75	V exit (m/s)	130.51

Table F1. LDV Repeatability Data (Wake Positions of -8 and 0 Degrees)

Wake Position	Depth	V-theta	V-z	V total	Mach	Flow Angle	V-theta	V-z	T stag (°F)	95.0
(degrees)	(mm)	(m/s)	(m/s)	(m/s)	Number	(degrees)	turb. int.	turb. int.	P static (inhg)	30.00
Coincidence /	0.01	88.5	24.5	91.8	0.261	74.5	8.06	5.59	P stag (inhg)	3.13
Coincidence /	0.06	89.4	24.2	92.6	0.263	74.9	8.14	5.75	Prat	0.9055
Coincidence /	0.18	92.1	24.5	95.3	0.271	75.1	7.76	5.74	M exit	0.379
Coincidence /	0.42	93.9	24.4	97.0	0.276	75.4	7.07	5.58	T exit (°K)	299.4
Coincidence /	0.89	98.3	24.6	101.3	0.288	76.0	6.25	5.25	a exit (m/s)	346.84
Coincidence /	1.78	101.0	24.4	103.9	0.295	76.4	4.82	4.97	a stag (m/s)	351.79
Coincidence /	3.34	103.0	25.5	106.1	0.302	76.1	4.33	4.70	V exit (m/s)	131.46
Wake Position	Depth	V-theta	V-z	V total	Mach	Flow Angle	V-theta	V-z	T stag (°F)	95.0
(degrees)	(mm)	(m/s)	(m/s)	(m/s)	Number	(degrees)	turb. int.	turb. int.	P static (inhg)	30.00
Random /	0.01	88.5	24.3	91.8	0.261	74.6	8.22	5.95	P stag (inhg)	3.13
Random /	0.06	89.9	24.6	93.2	0.265	74.7	7.68	6.06	Prat	0.9055
Random /	0.18	91.4	24.5	94.6	0.269	75.0	7.42	6.09	M exit	0.379
Random /	0.42	93.8	23.8	96.8	0.275	75.8	7.20	5.91	T exit (°K)	299.4
Random /	0.89	98.1	24.3	101.1	0.287	76.1	6.21	5.49	a exit (m/s)	346.84
Random /	1.78	101.0	24.4	103.9	0.295	76.4	4.61	4.66	a stag (m/s)	351.79
Random /	3.34	103.0	25.3	106.1	0.302	76.2	4.18	4.50	V exit (m/s)	131.46

Table F2. LDV Repeatability Data (Wake Position of 7 Degrees)

## APPENDIX G. COMPUTATIONAL DATA REDUCTION

Residual (fort.4) and P/P<sub>0</sub> (fort.7) files were produced from the solution (fort.3) files using the FORTRAN program **pxy.f**.

```

Program pxy.f
c *****
c pxy.f reads rvc3d files & writes ascii files for gnuplot
c unit 1 = input xyz file
c unit 3 = input q file
c unit 7 = output blade pressures on 5 k-planes
c unit 4 = output residual history
c *****
parameter(ni=150,nj=31,nk=65)
  real x(ni,nj,nk),y(ni,nj,nk),z(ni,nj,nk)
  real qq(5,ni,nj,nk),resd(5000,5)
  real pk(5),xk(5)
  dimension kk(5)
c k-values are hard-wired below (hub, 25%, 50%, 75%, tip)
  data kk/2,27,33,39,64/
c *****
c read grid coordinates
c *****
read(1,*)im,jm,km
  read(1,*)((x(i,j,k),i=1,im),j=1,jm),k=1,km),
  1 ((y(i,j,k),i=1,im),j=1,jm),k=1,km),
  2 ((z(i,j,k),i=1,im),j=1,jm),k=1,km)
c *****
c read restart file
c *****
read(3,*)imax,jmax,kmax
  read(3,*)fsmach,alpha,re,time
c
  icheck=iabs(im-imax)+iabs(jm-jmax)+iabs(km-kmax)
  if(icheck.ne.0)then
    write(6,610)im,jm,km,imax,jmax,kmax
    stop
  endif
c
  read(3,*)((qq(l,i,j,k),i=1,im),j=1,jm),k=1,km),l=1,5)
c
c additional residual data
  read(3,*)itl,iil,phdeg,ga,om,nres,dum,dum,dum,dum
  read(3,*)((resd(nr,l),nr=1,nres),l=1,5)
c *****
c ps/p0 output to unit 3
c *****
itr=im+1-itl
  ggm=ga*(ga-1.)
  j=1

```



```

c   normalize x by chord
do 7 l=1,5
  k=kk(l)
  xmin=x(im/2,j,k)
  xmax=x(itl,j,k)
  do 5 i=itl,itr
    xmin=min(xmin,x(i,j,k))
  5  xmax=max(xmax,x(i,j,k))
  chord=xmax-xmin
  do 7 i=itl,itr
    x(i,j,k)=(x(i,j,k)-xmin)/chord

    write(7,305)(kk(l),l=1,5)
    do 20 i=itl,itr
      do 10 l=1,5
        k=kk(l)
        pk(l)=ggm*(qq(5,i,j,k)-.5*(qq(2,i,j,k)**2+qq(3,i,j,k)**2
          1+qq(4,i,j,k)**2)/qq(1,i,j,k))
      10  xk(l)=x(i,j,k)
      20  write(7,300)i,(xk(l),pk(l),l=1,5)
c *****
c   residual history output to unit 4
c *****
write(4,310) 1,(resd(1,l),l=1,5)
do 30 j=2,nres
  it=10*(j-1)
  30  write(4,310)it,(resd(j,l),l=1,5)
c *****
300 format(i5,10f10.5)
305 format('  k=',5(17x,i3))
310 format(i5,5(1x,e10.3))
610 format(' ***** warning *****',/,
  1   ' im, jm, km, read from input',3i5,' do not match',/,
  2   ' im, jm, km, read from restart file',3i5)
stop
end

```

The **plane.f** program from Reference 6 was modified to compare two-dimensional experimental and computational Mach number and wake flow angle. Mach number, based on stagnation conditions, was redefined with respect to tangential and axial velocity components.

```

Program plane.f
c *****
c   Modified by Lt. William Donovan on 08 May 95 for Master's Thesis
c   plane.f reads rvc3d files & writes ascii files for plotting
c   exit plane mach number and flow angles
c   (2D comparison with experimental results)
c   unit 1 = input xyz file
c   unit 3 = input q file
c   unit 4 = output residual history
c   unit 7 = output (k #1) counter, theta, pt/p0,Mach, Angle
c   unit 8 = output (k #2) counter, theta, pt/p0,Mach, Angle
c   unit 9 = output (k #3) counter, theta, pt/p0,Mach, Angle
c   unit 10 = output (k #4) counter, theta, pt/p0,Mach, Angle
c   unit 11 = output (k #5) counter, theta, pt/p0,Mach, Angle
c
c *****
parameter(ni=150,nj=31,nk=65)
integer um
real x(ni,nj,nk),y(ni,nj,nk),z(ni,nj,nk), gama
real qq(5,ni,nj,nk),resd(5000,5),q1,q2,q3,q4,q5,m(5)
dimension kk(5),ang(5),theta(5),tt0(5),vtnd(5),vmd(5),
1 vznd(5),ptp0(5),deg(5)

c
c   k-values are hard-wired below (80.9%, 89.8%, 94.9%, 97.6%, 99.0%)
c   (use span.f to obtain %-k equivalency)
c   (k values of 61 and 64 obtained from a second program run)
c
c   data kk/41,45,49,53,57/
c *****
c   read grid coordinates
c *****
read(1,*)im,jm,km
  read(1,*)((x(i,j,k),i=1,im),j=1,jm),k=1,km),
  1 (((y(i,j,k),i=1,im),j=1,jm),k=1,km),
  2 (((z(i,j,k),i=1,im),j=1,jm),k=1,km)
c *****
c   read restart file
c *****
read(3,*)imax,jmax,kmax
  read(3,*)fsmach,alpha,re,time

c
c   icheck=iabs(im-imax)+iabs(jm-jmax)+iabs(km-kmax)
c   if(icheck.ne.0)then
c     write(6,610)im,jm,km,imax,jmax,kmax
c     stop
c   endif
c

```

```

      read(3,*)((((qq(1,i,j,k),i=1,im),j=1,jm),k=1,km),l=1,5)
c
c additional residual data
      read(3,*)itl,iil,phdeg,gama,om,nres,dum,dum,dum,dum
      read(3,*)((resd(nr,l),nr=1,nres),l=1,5)
c*****
c Pressure side of exit wake cut...
c*****
k=kk(3)
  i=1
  do 10 j=jmax,1,-1
    um=um+1
    do 15 l=1,5
      k=kk(l)
c
c Rename desired q values for use
c
      q1=qq(1,i,j,k)
      q2=qq(2,i,j,k)
      q3=qq(3,i,j,k)
      q4=qq(4,i,j,k)
      q5=qq(5,i,j,k)
c
c Compute pt/p0 ratio
c
      ptp0(l)=gama*(gama-1)*q5+(gama-gama**2/2)/q1*(q2**2+q3**2+q4**2)
c
c Compute T/Tt ratio and mach number
c
      tt0(l)=gama*(gama-1)/q1*(q5-0.5*(q2**2+q3**2+q4**2)/q1)
      m(l)=((1/tt0(l)-1)**2.0/(gama-1))**0.5
c
c Compute Theta and Non-D velocities theta, r and z
c
      theta(l)=asin(y(i,j,k)/(z(i,j,k)**2+y(i,j,k)**2)**0.5)
      vtnd(l)=-q4*sin(theta(l))+q3*cos(theta(l))
      vrnd(l)=q4*cos(theta(l))+q3*sin(theta(l))
      vznd(l)=q2
      m(l)=(sqrt(vtnd(l)**2+vznd(l)**2))/q1
      deg(l)=theta(l)*57.296
c
c Compute flow angle (degrees) using vtnd and vznd
c
      ang(l)=atan2(vtnd(l),vznd(l))*57.296
15 continue
      write(7,300)um,deg(1),ptp0(1),m(1),ang(1)
      write(8,300)um,deg(2),ptp0(2),m(2),ang(2)
      write(9,300)um,deg(3),ptp0(3),m(3),ang(3)
      write(10,300)um,deg(4),ptp0(4),m(4),ang(4)
      write(11,300)um,deg(5),ptp0(5),m(5),ang(5)
10 continue
c
c Repeat above on other side of wake cut at imax
c j=1 not used (duplicate location across wake)
c jmax not repeated (same location as j=31 at i=1)

```

```

c
jmaxm=jmax-1
i=im
do 30 j=2,jmaxm,1
um=um+1
do 25 l=1,5
k=kk(l)
q1=qq(1,i,j,k)
q2=qq(2,i,j,k)
q3=qq(3,i,j,k)
q4=qq(4,i,j,k)
q5=qq(5,i,j,k)
ptp0(l)=gama*(gama-1)*q5+(gama-gama**2/2)/q1*(q2**2+q3**2+q4**2)
tt0(l)=gama*(gama-1)/q1*(q5-0.5*(q2**2+q3**2+q4**2)/q1)
m(l)=(1/tt0(l)-1)*2/(gama-1))*0.5
theta(l)=asin(y(i,j,k)/(2*(i,j,k)**2+y(i,j,k)**2)**0.5)
vtnd(l)=-q4*sin(theta(l))+q3*cos(theta(l))
vmnd(l)=q4*cos(theta(l))+q3*sin(theta(l))
vznd(l)=q2
m(l)=(sqrt(vtnd(l)**2+vznd(l)**2))/q1
deg(l)=theta(l)*57.296
ang(l)=atan2(vtnd(l),vznd(l))*57.296
25 continue
write(7,300)um,deg(1),ptp0(1),m(1),ang(1)
write(8,300)um,deg(2),ptp0(2),m(2),ang(2)
write(9,300)um,deg(3),ptp0(3),m(3),ang(3)
write(10,300)um,deg(4),ptp0(4),m(4),ang(4)
write(11,300)um,deg(5),ptp0(5),m(5),ang(5)
30 continue
c *****
c residual history output to unit 4
c *****
write(4,310) 1,(resd(1,l),l=1,5)
do 40 j=2,nres
it=10*(j-1)
40 write(4,310)it,(resd(j,l),l=1,5)
c *****
300 format(i5,f8.3)
310 format(i5,5(1x,e10.3))
610 format(' ***** warning *****',/
1 ' im, jm, km, read from input',3i5,' do not match',/
2 ' im, jm, km, read from restart file',3i5)
stop
end

```



## APPENDIX H. MACH NUMBER AND FLOW ANGLE COMPARISONS

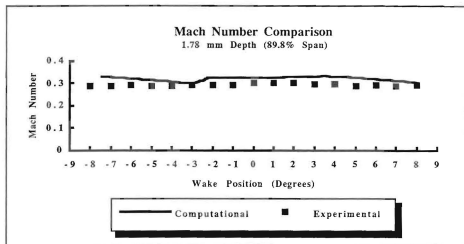


Figure H1. Mach Number Comparison At 1.78 mm Depth

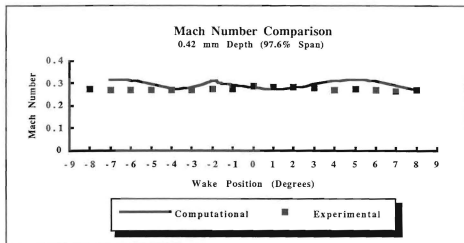


Figure H2. Mach Number Comparison At 0.42 mm Depth

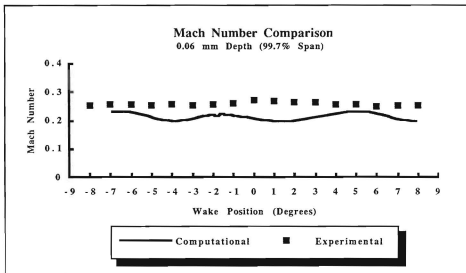


Figure H3. Mach Number Comparison At 0.06 mm Depth

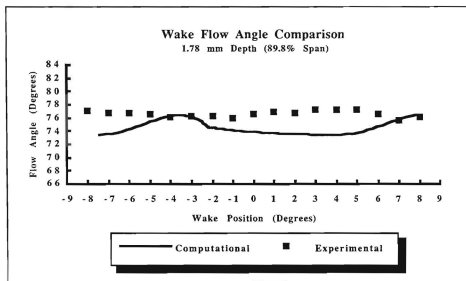


Figure H4. Flow Angle Comparison At 1.78 mm Depth

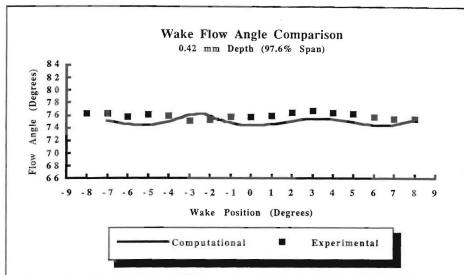


Figure H5. Flow Angle Comparison At 0.42 mm Depth

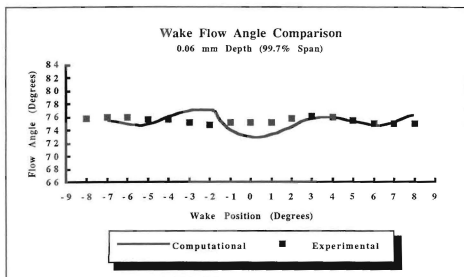


Figure H6. Flow Angle Comparison At 0.06 mm Depth





## APPENDIX I. CONVERGENCE HISTORY

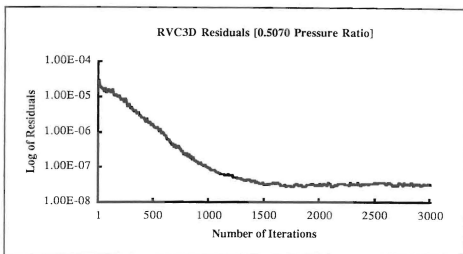


Figure 11. 0.5070 Pressure Ratio Convergence History

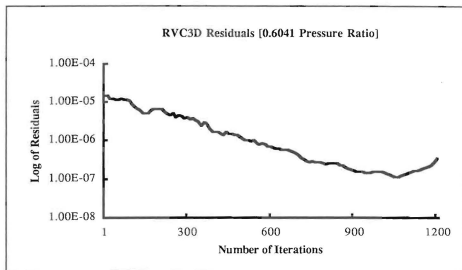


Figure 12. 0.6041 Pressure Ratio Convergence History

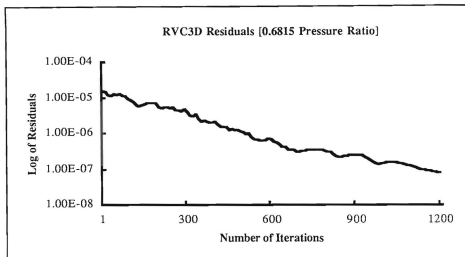


Figure I3. 0.6815 Pressure Ratio Convergence History

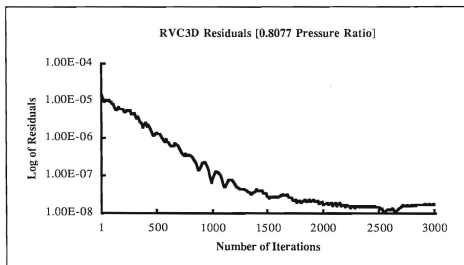


Figure I4. 0.8077 Pressure Ratio Convergence History

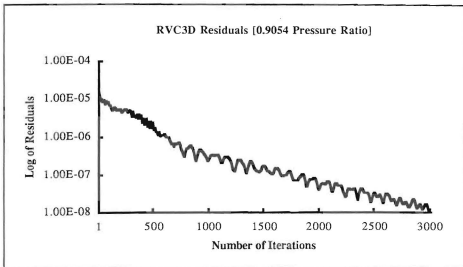


Figure I5. 0.9054 Pressure Ratio Convergence History



100

100

100

Program **mq.f** read a solution file (fort.3) and converted it into a multiple solution file format (fort.23) for use with PLOT3D.

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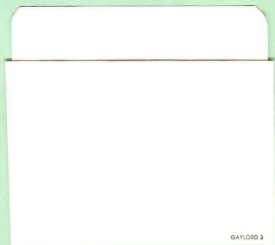
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